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Appendix A

A1 Fortran code for the PV/T TRNSYS component

```

SUBROUTINE TYPE262 (TIME,XIN,OUT,T,DTDT,PAR,INFO,ICNTRL,*)
C*****
C      PV/THERMAL COLLECTOR SUBROUTINE
C
C Author: Joe Coventry
C Date commenced:      9/7/2001 last modified: 10/2/2004
C
C This component is a detailed model of a concentrating PV/thermal collector.
C The model considers capacitance effects. The model is a detailed
C model, where the various quantities of heat transfer between cover,
C cells, plate, tube and fluid are calculated by iteratively solving the
C equations that physically describe the modes of heat transfer.
C
C*****
C      STANDARD TRNSYS DECLARATIONS
C          DOUBLE PRECISION XIN,OUT
C          PARAMETER (NIMAX=7,NPMAX=34,NO=13,ND=0)
C          INTEGER*4 INFO,ICNTRL,NP
C          REAL T,DTDT,PAR,TIME
C          DIMENSION XIN(NIMAX),OUT(NO),PAR(NPMAX),INFO(15)
C          CHARACTER*3 YCHECK(NIMAX),OCHECK(NO)
C
C      Declaration specific to this model
C
C      TRNSYS common variables needed for using the store S and the
C      simulation time step DELT
C
C          INCLUDE '!.INCLUDE\PARAM.INC'
C          COMMON /SIM/ TIME0,TIMEF,DELTA,IWARN
C          COMMON /STORE/ NSTORE,IAV,S(NUMSTR)
C          COMMON/LUNITS/LUR,LUW,IFORM,LUK
C
C      Maximum number of discrete elements (this can be changed)
C          PARAMETER (MAXCELLS=100)
C
C      External functions

```

EXTERNAL QTHFUNC1, CPWATER

C Some constants used in the model (temp in K at 0 DegC, Stefan-Boltzmann constant)
 DATA IUNIT/0/,TZERO/273.15/,SBC/20.41E-8/

C Declaration of parameters (commented below)

 INTEGER MODE
 INTEGER CELLS
 REAL REFEFF
 REAL REFTEMP
 REAL BETA
 REAL UNIFORMITY
 REAL LENGTH
 REAL WIDTH
 REAL REFLM
 REAL SHAPE
 REAL TRANSABS
 REAL COVERABS
 REAL EMIS
 REAL MASS
 REAL CP
 REAL TSTART
 REAL WGLASS
 REAL UCG
 REAL WCG
 REAL UCP
 REAL WCP
 REAL UPT
 REAL WPT
 REAL UINSUL
 REAL WINSUL
 REAL WCOVER
 REAL EMISCOV
 REAL CW0,CW1,CW2
 REAL TOL
 REAL HCTF
 REAL PERIM
 REAL XSAREA
 REAL FH

C Counters

 INTEGER J

C	Inputs used in the model (commented below)
	REAL TFI
	REAL FLOW
	REAL ID
	REAL TAMB
	REAL WIND
	REAL SHADE
	REAL DIRT
C	Variables used within the model (in order of appearance)
C	Capacitance of each node
	REAL CAPNODE
C	Sum of thermal output
	REAL QTHSUM
C	Sum of electrical output
	REAL QELECSUM
C	Sum of losses through insulation
	REAL QINSULSUM
C	Sum of convection losses from the glass
	REAL QCONVSUM
C	Sum of radation losses from the glass
	REAL QRADSUM
C	Sum of cell temperatures (used to give an average cell temp)
	REAL TCSUM
C	Sum of glass temperatures (used to give an average glass temp)
	REAL TGSUM
C	Sum of plate temperatures (used to give an average plate temp)
	REAL TPSUM
C	Sum of cell temperatures (used to give an average tube temp)
	REAL TTSUM
C	Sum of mid glass temperatures (used to give an average mid glass temp)
	REAL TGMIDSUM
C	Sum of insulation cover temperatures (used to give an average ins. cover temp)
	REAL TCOVSUM
C	Temperature of the inlet of the node
	REAL TFIN(MAXCELLS)
C	Average temperature of the fluid in the node
	REAL TBAR(MAXCELLS)
C	Temperature of the fluid in the node at the previous time step
	REAL TFINIT(MAXCELLS)

C Temperatre at the inlet of the node at the previous time step
REAL TFINPREV(MAXCELLS)

C Temperature of the fluid at the outlet of the node
REAL TFINAL(MAXCELLS)

C Upper and lower guesses for the bisection
REAL QTHUPPER,QTHLOWER

C Boolean flag to indicate if the solution is in the guessed bounds
LOGICAL SUCCESS

C Thermal output
REAL QTH

C Array of thermal outputs
REAL QTHARRAY(MAXCELLS)

C Electrical output
REAL QELEC

C Array of electrical output
REAL QELECARRAY(MAXCELLS)

C Convective heat loss from glass
REAL QCONV

C Array of convective heat loss from glass
REAL QCONVARRAY(MAXCELLS)

C Radiative heat loss from glass
REAL QRAD

C Array of radiative heat loss from glass
REAL QRADARRAY(MAXCELLS)

C Heat loss through the insulation
REAL QINS

C Array of heat loss through the insulation
REAL QINSARRAY(MAXCELLS)

C Tube temperature
REAL TT

C Array of tube temperatures
REAL TTARRAY(MAXCELLS)

C Plate temperature
REAL TP

C Array of plate temperatures
REAL TPARRAY(MAXCELLS)

C Cell temperature
REAL TC

C Array of cell temperatures
REAL TCARRAY(MAXCELLS)

C Glass surface temperature
REAL TG


```

C          Array of glass surface temperatures
          REAL TGARRAY(MAXCELLS)
C          Mid glass/silicone temperature
          REAL TGMID
C          Array of mid glass/silicone temperatures
          REAL TGMIDARRAY(MAXCELLS)
C          Insulation cover temperature
          REAL TCOVER
C          Array of insulation cover temperatures
          REAL TCOVARRAY(MAXCELLS)
C          Coefficients for the differential equation  $Dt/dt = AT + B$ 
          REAL AA, BB
C          Heat capacity of the water
          REAL CPF

C          Common block variables used in functions
COMMON/PVTPARAMS/ID,WIDTH,LENGTH,CELLS,REFLM,SHAPE,SHADE,DIRT,CW0,
1          CW1,CW2,WIND,MODE,TF,XSAREA,PERIM,FLOW,FH,HCTF,TT,TP,UPT,
1          WPT,UINSUL,WINSUL,TAMB,SBC,EMISCOV,WCOVER,TCOVER,QINS,
1          QCP,UCP,WCP,QELEC,REFEFF,BETA,REFTEMP,TC,QABSCELLS,
1          TRANSABS,QCG1,TGMID,UCG,WCG,QABSGLASS,COVERABS,QCG2,TG,
1          QCONV,WGLASS,QRAD,EMIS,UNIFORMITY

C-----
C          If it is the very first call of the simulation then continue,
C          with these initialisation things otherwise go down to parameters.

          IF (INFO(7).GE.0) GO TO 10

C          Set this to number of outputs
          INFO(6)=NO

C          Set to 1 as routine depends on passage of time
          INFO(9)=1

C          Set the common store size to the maximum number of cells
          INFO(10)=2*MAXCELLS

C          Check that the user has provided the right number of inputs, outputs
C          and derivatives

          CALL TYPECK(1,INFO,NIMAX,NPMAX,ND)

```

C Set the input and output variable types

```

DATA YCHECK/'TE1','MF1','IR1','TE1','VE1','DM1','DM1'/
DATA OCHECK/'TE1','MF1','PW1','PW1','PW1','PW1','PW1',
1      'TE1','TE1','TE1','TE1','TE1','TE1'/
CALL RCHECK(INFO,YCHECK,OCHECK)

```

C Set the first storage place in the middle of the allocated variables
 ISTORE=INFO(10)

C-----

C Get the values of the parameter for this component (only once)

```

10 IF(INFO(1) .EQ. IUNIT) GO TO 30
   IUNIT=INFO(1)

```

C First do common parameters

C MODE - whether or not the fluid convection coefficient is calculated
 explicitly 1 = no, 2 = yes

```
MODE=INT(PAR(1)+0.1)
```

C CELLS - number of nodes in the receiver is divided into along its length

```
CELLS=INT(PAR(2)+0.1)
```

C REFEFF - the reference efficiency of an encapsulated the solar cell

```
REFEFF=PAR(3)
```

C REFTEMP - the reference temperature for calculation of solar cell efficiency

```
REFTEMP=PAR(4)+TZERO
```

C BETA - the coefficient relating cell efficiency and temperature

```
BETA=PAR(5)
```

C UNIFORMITY - Scaling factor to account for the drop in electrical performance of the
 C receiver due to both longitudinal and transverse non-uniform radiation and temperature

```
UNIFORMITY = PAR(6)
```

C LENGTH - Length of the troughs

```
LENGTH = PAR(7)
```

C WIDTH - Width of the mirror (ie unshaded bit)

WIDTH=PAR(8)

C REFLM - Reflectivity of the mirror
REFLM=PAR(9)

C SHAPE - Optical accuracy of the mirror (1 = perfect)
SHAPE=PAR(10)

C TRANSABS - the transmission absorption coefficient for the cells
TRANSABS = PAR(11)

C COVERABS - the fraction of light passing through the cover that is absorbed
COVERABS = PAR(12)

C EMIS - the emissivity of the surface of the cell encapsulation
EMIS=PAR(13)

C MASS - mass of the receiver and fluid combined
MASS=PAR(14)+0.01

C CP - thermal capacitance of the receiver and fluid combined
CP=PAR(15)

C TSTART - initial temperature of the receiver and fluid
TSTART=PAR(16)+TZERO

C WGLASS - Width of glass cover for the purposes of calculating convection
and radiation losses
WGLASS=PAR(17)

C UCG - U value between the cells and the glass
UCG=PAR(18)

C WCG - Width of the connection between cells and glass for heat transfer purposes
WCG = PAR(19)

C UCP - U value between the cells and the plate
UCP=PAR(20)

C WCP - Width of interface between the cells and the plate
WCP = PAR(21)

- C UPT - U value between the plate and tube
UPT=PAR(22)
- C WPT- Width of the interface between plate and tube
WPT = PAR(23)
- C UINSUL - U value for the insulation
UINSUL=PAR(24)
- C WINSUL - Perimeter length for calculating insulation losses
WINSUL = PAR(25)
- C WCOVER - Perimeter length for calculating convection and radiation losses
C from the insulation cover
WCOVER = PAR(26)
- C EMISCOV - Emissivity of the insulation cover
EMISCOV = PAR(27)
- C CW0, CW1, CW2 - Coefficients for calculation of convective heat loss from
C all outer surfaces of the receiver. $h_c = CW0 + CW1 * \text{Wind speed} + CW2 * \text{Wind speed}^2$
CW0=PAR(28)
CW1=PAR(29)
CW2=PAR(30)
- C TOL - tolerance of the iterative calculation of Qth
TOL=PAR(31)
- C-----
- C Now do parameters specific to modes of operation
- C Firstly, the input mode determines whether the convection
C coefficient should be calculated explicitly or entered
C as a parameter
- IF (MODE.EQ.1) THEN
- C HCTF - coefficient of convection for the fluid set as a parameter
HCTF=PAR(32)/CELLS
- ELSEIF (MODE.EQ.2) THEN

```

C          PERIM - Wetted perimeter of the flow passage in the tube
              PERIM=PAR(32)

C          XSAREA - cross-sectional area of the flow passage in the tube
              XSAREA = PAR(33)

C          FH - Correction for Nusselt number to account for the internal fins
C          (ie. Using the Carnavos relation gives 0.74 for the CHAPS receiver)
              FH=PAR(34)

          ENDIF

C-----
C          Set the intial values for fluid temperature in the receiver

              IF(INFO(7).EQ.-1) THEN
                  DO 20 J=1,CELLS
                      S(ISTORE+(J-1))=TSTART
20              CONTINUE
              Return 1
              ENDIF

C-----
30          CONTINUE

C          Set the storage place for this particular component
              ISTORE=INFO(10)

C          Get the common values for the inputs for this component

C          TFI - Inlet fluid temperature (degC)
              TFI=XIN(1)+TZERO

C          FLOW - Flow rate (kg/hr)
              FLOW=XIN(2)

C          ID - Direct beam radiation (kJ/hr.m2)
              ID=XIN(3)

C          TAMB- Ambient temperature (degC)
              TAMB=XIN(4)+TZERO

```

```

C          WIND - Wind speed (m/s)
              WIND=XIN(5)

C          SHADE - fraction of the mirrors that are unshaded by adjacent mirrors
C              1 = no shading, 0 = full shading
              SHADE=XIN(6)

C          DIRT - Measure of cleanliness of the mirrors (1 = perfectly clean)
              DIRT=XIN(7)

C-----
C          Retrieve the outlet temps from the previous time step
              DO 40 J=1,CELLS
                  TFINIT(J)= S(ISTORE+(J-1))
40          CONTINUE

C          Retrieve inlet temps from the previous time step
              DO 45 J=1,CELLS
                  TFINPREV(J)= S(ISTORE+(J-1)+CELLS)
45          CONTINUE

C-----
C          Thermal performance
C-----

C          Set the capacitance for each cell node

              CAPNODE=(MASS*CP)/CELLS

C          Initialise energy sums for this timestep
              QTHSUM=0
              QELECSUM=0
              QINSULSUM=0
              QCONVSUM=0
              QRADSUM=0

C          Initialise temp sums (to be used to calculate mean temperature)
              TCSUM=0
              TGSUM=0
              TPSUM=0
              TTSUM=0
              TGMIDSUM=0

```

```

TCOVSUM=0

C      Set up the loop for the cells
50     DO 60 J=1,CELLS

C      Set the inlet fluid temp

          IF (J.EQ.1) THEN
              TFIN(J)=TFI
          ELSE
              TFIN(J)=TBAR(J-1)
          ENDIF

C      Set fluid temp as average temp in the element at the previous time step
          TF=(TFINPREV(J)+TFINIT(J))/2

C      The following uses the Bisection Algorithm routine to solve the
C      simultaneous equations describing this system.

C      For a concentrating collector

C      Set the lower limit for the bisection (zero heat transfer)
          QTHLOWER=0

C      Set the upper limit for the bisection (extraterrestrial radiation)
          QTHUPPER=4760*WIDTH*LENGTH/CELLS

C      Check if the limits bound the solution by using a bracketing algorithm
          CALL ZBRAC(QTHFUNC1,QTHLOWER,QTHUPPER,SUCCESS)

C      If the solution is bounded, then proceed to calculate the thermal heat
C      transfer, Qth

          IF (SUCCESS) THEN CALL BISECTION(QTHFUNC1,QTHLOWER,QTHUPPER,TOL,QTH)

C      Otherwise there is a problem
          END IF

C      Store energy transfer values in arrays
          QTHARRAY(J)=QTH
          QINSARRAY(J)=QINS

```

```

QELECARRAY(J)=QELEC
QRADARRAY(J)=QRAD
QCONVARRAY(J)=QCONV

```

C Store various temperatures in arrays

```

TTARRAY(J)=TT
TPARRAY(J)=TP
TCARRAY(J)=TC
TGARRAY(J)=TG
TGMIDARRAY(J)=TGMID
TCOVARRAY(J)=TCOVER

```

C Find heat capacity of the fluid

```

CPF = CPWATER(TF)/1000

```

C Set up the differential equation for the collector in the form $dT/dt=AT+B$

```

BB=QTH/CAPNODE+FLOW*CPF*TFIN(J)/CAPNODE
AA=-FLOW*CPF/CAPNODE

```

C Solve the differential equation analytically.

```

IF(AA.EQ.0.) THEN
    TFINAL(J)=TFINIT(J)+BB*DELT
    TBAR(J)=TFINIT(J)+BB*DELT/2.
ELSE
    TFINAL(J)=TFINIT(J)*(EXP(AA*DELT))
    +BB/AA*(EXP(AA*DELT))-BB/AA
    TBAR(J)=1./AA/DELT*(TFINIT(J)+BB/AA)*
    ((EXP(AA*DELT))-1.)-BB/AA
ENDIF

```

C Update energy sums and temp sums

```

QTHSUM=QTHSUM+QTH
QELECSUM=QELECSUM+QELEC
QINSULSUM=QINSULSUM+QINS
QCONVSUM=QCONVSUM+QCONV
QRADSUM=QRADSUM+QRAD
TGSUM=TGSUM+TG

```



```

TTSUM=TTSUM+TT
TPSUM=TPSUM+TP
TCSUM=TCSUM+TC
TGMIDSUM = TGMIDSUM+TGMID
TCOVSUM = TCOVSUM + TCOVER

60      CONTINUE

C      SET THE OUTPUTS
70      CONTINUE

C      Outlet fluid temperature (degC)
      OUT(1)=TBAR(CELLS)-TZERO

C      Outlet flow rate (kg/hr)
      OUT(2)=FLOW

C      Electrical output (kJ/hr)
      OUT(3)=QELECSUM

C      Thermal output (kJ/hr)
      OUT(4)=QTHSUM

C      Insulation losses (kJ/hr)
      OUT(5)=QINSULSUM

C      Convection losses (kJ/hr)
      OUT(6)=QCONVSUM

C      Radiation losses (kJ/hr)
      OUT(7)=QRADSUM

C      Mean cell temperature (degC)
      OUT(8)=TCSUM/CELLS-TZERO

C      Mean glass temperature (degC)
      OUT(9)=TGSUM/CELLS-TZERO

C      Mean plate temperature (degC)
      OUT(10)=TPSUM/CELLS-TZERO

C      Mean tube temperature (degC)
      OUT(11)=TTSUM/CELLS-TZERO

C      Mean mid glass/silicone temperature (degC)
      OUT(12)=TGMIDSUM/CELLS-TZERO

C      Mean insulation cover temperature (degC)
      OUT(13)=TCOVSUM/CELLS-TZERO

C      Enter final temp data into store for next timestep
      DO 80 J=1,CELLS
          S(ISTORE+(J-1))=TFINAL(J)
80      CONTINUE

```

```

C          Enter inlet temp data into store for next timestep
              DO 90 J=1,CELLS
                  S(ISTORE+(J-1)+CELLS)=TFIN(J)
90          CONTINUE

              RETURN 1
              END

```

```

C-----
REAL FUNCTION QTHFUNC1(QTHERMAL)

```

```

C          This function calculates the equations that
C          describe the collector.
C          The physically meaningless objective function
C          is returned.
C          QTHERMAL is the guess,
C          TF is the known fluid temp that varies for each node

```

```

c          External functions

```

```

          EXTERNAL VISCOSITY,CONDUCTIVITY,CPWATER, DENSITY

```

```

C          Internal variables

```

```

C          Radiation incident on the receiver
          REAL QSUN
C          Coefficient of convection for air past the receiver
          REAL HCCONV
C          Viscosity of water at atmospheric pressure
          REAL MU_W
C          Conductivity of water at atmospheric pressure
          REAL K_W
C          CP value of water at atmospheric pressure
          REAL CP_W
C          Density of water at atmospheric pressure
          REAL DENSITY_W
C          Prandtl number of water at atmospheric pressure
          REAL PR
C          Hydraulic diameter of receiver
          REAL DH

```

```

C          Bulk fluid velocity through receiver
          REAL VF
C          Reynolds number
          REAL RE
C          Nusselt number
          REAL NU
C          Coefficient of convection for water in the tube
          REAL HC
C          Short term variable for heat transfer
          REAL H1, H2, H3
C          Estimate of average of insulation cover temp and ambient temp
          REAL TMEAN

C          Common block variables used in functions
COMMON/PVTPARAMS/ID,WIDTH,LENGTH,CELLS,REFLM,SHAPE,SHADE,DIRT,CW0,
1          CW1,CW2,WIND,MODE,TF,XSAREA,PERIM,FLOW,FH,HCTF,TT,TP,UPT,
1          WPT,UINSUL,WINSUL,TAMB,SBC,EMISCOV,WCOVER,TCOVER,QINS,
1          QCP,UCP,WCP,QELEC,REFEFF,BETA,REFTEMP,TC,QABSCELLS,
1          TRANSABS,QCG1,TGMID,UCG,WCG,QABSGLOSS,COVERABS,QCG2,TG,
1          QCONV,WGLASS,QRAD,EMIS,UNIFORMITY

C          Re-declaration of parameters (from main program)
          INTEGER MODE
          INTEGER CELLS
          REAL REFEFF
          REAL REFTEMP
          REAL BETA
          REAL UNIFORMITY
          REAL LENGTH
          REAL WIDTH
          REAL REFLM
          REAL SHAPE
          REAL TRANSABS
          REAL COVERABS
          REAL EMIS
          REAL MASS
          REAL CP
          REAL TSTART
          REAL WGLASS
          REAL UCG
          REAL WCG
          REAL UCP

```

REAL WCP
REAL UPT
REAL WPT
REAL UINSUL
REAL WINSUL
REAL WCOVER
REAL EMISCOV
REAL CW0,CW1,CW2
REAL HCTF
REAL PERIM
REAL XSAREA
REAL FH

C Inputs used in the model

REAL TFI
REAL FLOW
REAL ID
REAL TAMB
REAL WIND
REAL SHADE
REAL DIRT

C Variables used within the model

C Electrical output

REAL QELEC

C Convective heat loss from glass

REAL QCONV

C Radiative heat loss from glass

REAL QRAD

C Heat loss through the insulation

REAL QINS

C Tube temperature

REAL TT

C Plate temperature

REAL TP

C Cell temperature

REAL TC

C Glass surface temperature

REAL TG

C Mid glass/silicone temperature

```

REAL TGMID
C      Insulation cover temperature
REAL TCOVER

C      Calculate the radiation incident on the cells
QSUN=ID*WIDTH*LENGTH/CELLS*REFLM*SHAPE*SHADE*DIRT

C      Calculate the convection coefficient due to wind
HCCONV=CW0+CW1*WIND+CW2*WIND*WIND

C      Calculate convection coeff in the pipe if required
IF (MODE.EQ.2) THEN

C      Calculate saturated water properties at atmospheric pressure
C      based on curve fits from Incropera and De Witt, 'Fundamentals
C      of Heat and Mass Transfer'

      MU_W=VISCOSITY(TF)
      K_W=CONDUCTIVITY(TF)
      CP_W=CPWATER(TF)
      DENSITY_W = DENSITY(TF)

C      Prandtl number
      PR = CP_W*MU_W/K_W

C      DH - Hydraulic diameter - defined as 4 x Flow cross sectional area / wetted perimeter
      DH = 4*XSAREA/PERIM

C      VF - Fluid velocity (m/s) (also convert kg/hr to m3/s)
      VF = FLOW/3600000/XSAREA

C      Reynolds number (based on hydraulic diameter)
      RE = DENSITY_W*VF*DH/MU_W

C      The accuracy of this method is not so good for Reynolds numbers below about 4000
C      Experimentally, it was found that using unmodified Dittus-Boelter seemed to produce
C      a decent fit down to about Re = 1000 (reasons given in thesis). Below 1000 the
C      Nusselt is set to 5 but this is only so the model doesn't crash. Not recommended.

      IF (RE.LT.1000) THEN

          NU = 10 ! Rough value to stop model crashing when there is no flow

```

```

ELSEIF ((RE.GE.1000).AND.(RE.LE.4000)) THEN

C          Use Dittus - Boelter to calculate Nu
          NU = 0.023*RE**0.8*PR**0.4

ELSE

C          Use Dittus - Boelter to calculate Nu
          NU = 0.023*RE**0.8*PR**0.4

C          Adjust to account for the internal fins
          NU = NU * FH

ENDIF

C          Calculate coefficient of convection based on the hydraulic diameter
          HC = NU*K_W/DH

C          Convert units back from W/m2.K to kJ/hr.m2.K
          HC = HC * 3.6

C          Multiply HC through but the wetted surface area
          HCTF = HC*PERIM*LENGTH/CELLS

END IF

C          Write equations that describe system without air gap

C          Calculate tube temperature
          TT=QTHERMAL/HCTF+TF

C          Calculate plate temperature
          TP=QTHERMAL/(UPT*LENGTH/CELLS*WPT)+TT

C          Calculate cover temperature by working out heat transfer coeffs for
C          a) conduction through insulation
          H1 = UINSUL*LENGTH/CELLS*WINSUL

C          b) radiation loss from the cover
C          estimate of mean temp between cover and ambient
          TMEAN = TAMB + 5
          H2 = 4*SBC*EMISCOV*WCOVER*LENGTH/CELLS*TMEAN**3

C          c) convection loss from the cover
          H3 = HCCONV*WCOVER*LENGTH/CELLS

C          Now put it together to calculate temperature of the cover

```

$$TCOVER = (H1*TP+(H2+H3)*TAMB)/(H1+H2+H3)$$

C Calculate losses through the insulation

$$QINS = H1 * (TP-TCOVER)$$

C Calculate heat transfer from the cells to the plate

$$QCP = QTHERMAL + QINS$$

C Calculate the temperature of the cells

$$TC = QCP/(UCP*LENGTH/CELLS*WCP)+TP$$

C Calculate the electrical output

$$QELEC=QSUN*REFEFF*EXP(BETA*(TC-REFTEMP))*UNIFORMITY$$

C Calculate the solar absorption by the cells

$$QABSCELLS = QSUN*TRANSABS*(1-COVERABS)$$

C Calculate QCG1

$$QCG1=QABSCELLS-QELEC-QCP$$

C Calculate temperature at glass midpoint

$$TGMID = TC-QCG1/(2*UCG*WCG*LENGTH/CELLS)$$

C Calculate the solar absorption in the cover glass/silicone

$$QABSGLASS = QSUN*TRANSABS*COVERABS$$

C Calculate QCG2

$$QCG2=QCG1+QABSGLASS$$

C Calculate the glass temperature

$$TG = TGMID - QCG2/(2*UCG*WCG*LENGTH/CELLS)$$

C Calculate convection losses from the glass cover

$$QCONV=HCCONV*WGLASS*LENGTH/CELLS*(TG-TAMB)$$

C Calculate radiation losses from the glass cover

$$QRAD=SBC*EMIS*WGLASS*LENGTH/CELLS*(TG**4-TAMB**4)$$

C Calculate objective function that is to be solved

$$QTHFUNC1=QCG2-(QRAD+QCONV)$$

END

C-----

C Bisection method

RECURSIVE SUBROUTINE BISECTION(FUNC,X1,X2,XACC,XROOT)

C Using bisection, finds the root of a function FUNC known to lie between
 C X1 and X2. The root, returned as XROOT, will be refined until its accuracy
 C is +/- XACC

COMMON/LUNITS/LUR,LUW,IFORM,LUK

REAL FMID,F,XROOT,DX,XMID

PARAMETER (JMAX=50)

FMID=FUNC(X2)

F=FUNC(X1)

IF(F*FMID.GE.0.) THEN

WRITE(LUW,'(A,I3,A,I3,A)') ' ERROR at Type 262 (Unit',unit,

1 '): Bisection method out of range'

CALL MYSTOP(1001)

ENDIF

IF(F.LT.0.) THEN

XROOT=X1

DX=X2-X1

ELSE

XROOT=X2

DX=X1-X2

ENDIF

DO 15000 J=1,JMAX

DX=DX*.5

XMID=XROOT+DX

FMID=FUNC(XMID)

IF(FMID.LT.0.) XROOT=XMID

IF(ABS(DX).LT.XACC .OR. FMID.EQ.0.) RETURN

15000 CONTINUE

END

C-----

SUBROUTINE ZBRAC (FUNC,X1,X2,SUCCESS)

C Given a function FUNC and an initial guessed range X1 to X2, the routine
 C expands the range geometrically until a root is bracketed by the returned
 C values of X1 and X2.

PARAMETER (FACTOR=1.6,NTRY=50)

LOGICAL SUCCESS

IF(X1.EQ.X2)PAUSE 'You have to guess an initial range'

F1=FUNC(X1)

F2=FUNC(X2)

SUCCESS =.TRUE.

DO 16000 J=1,NTRY

IF (F1*F2.LT.0.)RETURN

IF(ABS(F1).LT.ABS(F2))THEN

X1=X1+FACTOR*(X1-X2)

F1=FUNC(X1)

ELSE

X2=X2+FACTOR*(X2-X1)

F2=FUNC(X2)

END IF

16000 CONTINUE

SUCCESS=.FALSE.

RETURN

END

C-----

C

c WATER PROPERTIES - from curve fits to Incropera and De Witt, Saturated water

C at atmospheric pressure

c

Real FUNCTION CPWATER(T)

real T

CPWATER=0.0000032759702*T**4 - 0.0043807394*T**3 + 2.2005092*T**2

1 - 491.60617*T + 45358.904

end

Real FUNCTION CONDUCTIVITY(T)

real T

CONDUCTIVITY=-0.48064+0.0058471*T-0.0000073317*T**2

end

REAL FUNCTION VISCOSITY(T)

real T

VISCOSITY=0.00000000028665*T**4 - 0.000000039376*T**3

1 + 0.000020328*T**2 - 0.0046803*T + 0.40639

END

REAL FUNCTION DENSITY(T)

real T

T = T - 273.15

DENSITY=0.0000149*T**3 - 0.0057637*T**2 + 0.0063843*T + 1000.2418616

T = T + 273.15

END

A2 Fortran code for the End Loss TRNSYS component

```

SUBROUTINE TYPE123 (TIME,XIN,OUT,T,DTDT,PAR,INFO,ICNTRL,*)
C*****
C Object: End Loss Modifier
C IISiBat Model: EndLoss
C
C Author: Joe Coventry
C Editor: Joe Coventry
C Date: 20/6/2000 last modified: 20/6/2000
C
C Calculates the end losses from a parabolic mirror
C*****

C      STANDARD TRNSYS DECLARATIONS
      DOUBLE PRECISION XIN,OUT
      INTEGER NI,NP,ND,NO
      PARAMETER (NI=1,NP=3,NO=1,ND=0)
      INTEGER*4 INFO,ICNTRL
      REAL T,DTDT,PAR,TIME
      DIMENSION XIN(NI),OUT(NO),PAR(NP),INFO(15)
      CHARACTER*3 YCHECK(NI),OCHECK(NO)
      DATA RDCONV/0.017453/

C-----

C      IF ITS THE FIRST CALL TO THIS UNIT, DO SOME BOOKKEEPING
      IF (INFO(7).GE.0) GO TO 100

C      FIRST CALL OF SIMULATION, CALL THE TYPECK SUBROUTINE TO CHECK THAT THE
C      USER HAS PROVIDED THE CORRECT NUMBER OF INPUTS, PARAMETERS, AND
      DERIVS

      INFO(6)=NO
      INFO(9)=1
      CALL TYPECK(1,INFO,NI,NP,ND)
      RETURN 1

C      END OF THE FIRST ITERATION BOOKKEEPING

C-----

```

C GET THE VALUES OF THE PARAMETERS FOR THIS COMPONENT
100 CONTINUE

Trough_Length=PAR(1)

Trough_Width=PAR(2)

Focal_Length=PAR(3)

C GET THE VALUES OF THE INPUTS TO THIS COMPONENT
Incidence_Angle=XIN(1)

C-----

C End losses calculated based on the geometry of the trough and
C the incidence angle of light on a single axis tracking trough

AAA = Focal_Length/Trough_Length

BBB = (Trough_Width**2)/(48*Focal_Length**2)

End_Loss_Factor = 1-AAA*(1+BBB)*tan(Incidence_Angle*RDCONV)

C-----

C SET THE OUTPUTS
200 CONTINUE

C End Loss Factor
OUT(1)=End_Loss_Factor

RETURN 1

END

A3 Fortran code for the Fin-tube TRNSYS component

```

SUBROUTINE TYPE273 (TIME,XIN,OUT,T,DTDT,PAR,INFO,ICNTRL,*)
C*****
C Object: Improved fin tube model
C IISiBat Model: Type 273
C
C Author: Joe Coventry
C Editor:
C Date: 6/8/2002 last modified: 24/5/2004
C
C          STANDARD TRNSYS DECLARATIONS
C          DOUBLE PRECISION XIN,OUT
C          INTEGER NIMAX,NPMAX,ND,NO
C          PARAMETER (NIMAX=4,NPMAX=11,NO=2,ND=0)
C          INTEGER*4 INFO,ICNTRL
C          REAL T,DTDT,PAR,TIME
C          DIMENSION XIN(NIMAX),OUT(NO),PAR(NPMAX),INFO(15)
C          CHARACTER*3 YCHECK(NIMAX),OCHECK(NO)

C          My declarations

C          Some constants used in the model
C          DATA CPF/4.18/IUNIT/0/

C          PARAMETERS

C          Length of the finned tube
C          REAL L

C          Number of fins
C          REAL NFINS

C          Area of the fins
C          REAL AFINS

C          Diameter of the tube
C          REAL DTUBE

c          Thickness of the fins
C          REAL TFINS

C          Conductivity of the fins
C          REAL KFINS

C          Number of discrete elements
C          REAL NODES

```

C Heat capacity of the fluid
 REAL CPF

C Heat capacity of the pipe
 REAL CPP

C Mass of the fluid
 REAL MASSF

C Mass of the pipe
 REAL MASSP

C INPUTS

C Inlet temperature to the fin-tube
 REAL TIN

C Ambient temperature
 REAL TAMB

C Flow rate (kg/hr)
 REAL FLOW

C Wind speed (m/s)
 REAL WS

C VARIABLES IN CALCULATION

C Energy transfer through an element of the fin-tube
 EXTERNAL QTHFIN

C Maximum number of discrete elements
 PARAMETER (MAXNODES=100)

C Sum of thermal energy transfer through the fin-tube
 REAL QTHSUM

C Fluid temperature
 REAL TF

C Temperature of the inlet of the node
 REAL TFIN(MAXNODES)

C Average temperature of the fluid in the node
 REAL TBAR(MAXNODES)

C Temperature of the fluid in the node at the previous time step
 REAL TFINIT(MAXNODES)

C Temperature of the fluid at the outlet of the node
 REAL TFINAL(MAXNODES)

C Boolean indicator of success of bisection method
 LOGICAL SUCCESS

C Need to include this file
 INCLUDE '..\INCLUDE\PARAM.INC'

```

C          Common variables to be used in other subroutines
          COMMON/FINPARAMS/ L, NFINS, AFINS, DTUBE, TFINS, KFINS,
1          TIN, TAMB, FLOW, WS, TF, NODES
          COMMON /SIM/ TIME0,TIMEF,DELT,IWARN
          COMMON /STORE/ NSTORE,IAV,S(NUMSTR)
          COMMON/LUNITS/LUR,LUW,IFORM,LUK

```

C-----

```

C          If it is the very first call of the simulation then continue,
C          with these initialisation things otherwise go down to parameters.

```

```

          IF (INFO(7).GE.0) GO TO 10

```

```

C          Set this to number of outputs
          INFO(6)=NO

```

```

C          Set to 1 as routine depends on passage of time
          INFO(9)=1

```

```

C          Set the common store size to the maximum number of cells
          INFO(10)=MAXNODES

```

```

C          Check that the user has provided the right number of inputs, outputs
C          and derivatives

```

```

          CALL TYPECK(1,INFO,NIMAX,NPMAX,ND)

```

```

C          Set the first storage place in the middle of the allocated variables
          ISTORE=INFO(10)

```

C-----

```

C          Get the values of the parameter for this component (only once)

```

```

10         IF(INFO(1) .EQ. IUNIT) GO TO 30

```

```

          IUNIT=INFO(1)

```

```

          L=PAR(1)
          NFINS=PAR(2)
          AFINS=PAR(3)
          DTUBE=PAR(4)
          TFINS=PAR(5)
          KFINS=PAR(6)
          NODES=INT(PAR(7)+0.001)

```

```

      CPP=PAR(8)
      MASSP=PAR(9)+0.01
      CPF=PAR(10)
      MASSF=PAR(11)+0.01

```

```

C      Set the initial values for fluid temperature, equivalent the average temp
C      if the collector was left for a while with no losses after initial conditions.
C      Also set initial values of QOLD to zero
      IF(INFO(7).EQ.-1) THEN
          TSTART=300
          DO 20 J=1,NODES
              S(ISTORE+(J-1))=TSTART
20          CONTINUE
      Return 1
      ENDIF

```

C-----

```

30      CONTINUE

```

```

C      Set the storage place for this particular component
          ISTORE=INFO(10)

```

```

C      GET THE VALUES OF THE INPUTS TO THIS COMPONENT
          TIN=XIN(1)+273.13
          FLOW=XIN(2)
          TAMB=XIN(3)+273.13
          WS=XIN(4)

```

```

C      Retrieve initial fluid temps and QOLD from the s-array
          DO 40 J=1,NODES
              TFINIT(J)= S(ISTORE+(J-1))
40          CONTINUE

```

C-----

```

C      Thermal performance

```

C-----

```

      IF (FLOW.EQ.0.) GO TO 70

```

```

C      Set the capacitance for each cell node

```

```

      CAPNODE=(MASSF*CPF+MASSP*CPP)/NODES
      QTHSUM = 0

```



```

50      DO 60 J=1,NODES

C      Set the inlet fluid temp
          IF (J.EQ.1) THEN
              TFIN(J)=TIN
          ELSE
              TFIN(J)=TBAR(J-1)
          ENDIF

C      Set fluid temp to original fluid temp of the node (at prev. time step)
          TF=TFINIT(J)

c      Note: because of this assumption the model may be unstable in the intial time
C      steps if mass flow is low, or the heat exchanger is a lot longer than it needs
C      to be.

C      The following uses the Bisection Algorithm routine to solve the
C      simultaneous equations describing this system.

C      Make guess of values
          QTHLOWER=0
          QTHUPPER=4760*L/NODES
          CALL ZBRAC3(QTHFIN,QTHLOWER,QTHUPPER,SUCCESS)
          CALL BISECTION3(QTHFIN,QTHLOWER,QTHUPPER,0.1,QTH)

C      Set up the differential equation for the collector in the form dT/dt=AT+B

          BB=-QTH/CAPNODE+FLOW*CPF*TFIN(J)/CAPNODE
          AA=-FLOW*CPF/CAPNODE

C      Solve the differential equation analytically.

          IF(AA.EQ.0.) THEN
              TFINAL(J)=TFINIT(J)+BB*DELTA
              TBAR(J)=TFINIT(J)+BB*DELTA/2.
          ELSE
              TFINAL(J)=TFINIT(J)*(EXP(AA*DELTA))
1          +BB/AA*(EXP(AA*DELTA))-BB/AA
              TBAR(J)=1./AA/DELTA*(TFINIT(J)+BB/AA)*
1          ((EXP(AA*DELTA))-1.)-BB/AA

```

```

                ENDIF

C              Update energy sums and temp sums
                QTHSUM=QTHSUM+QTH

60             CONTINUE

C              SET THE OUTPUTS
70            CONTINUE
C              Outlet fluid temperature
                IF (FLOW.EQ.0) THEN
                    OUT(1) = TAMB-273.13
                    DO 75 J=1,NODES
                        S(ISTORE+(J-1))=TFINIT(J)
75             CONTINUE
                ELSE
                    OUT(1)=TBAR(NODES)-273.13
                    DO 80 J=1,NODES
                        S(ISTORE+(J-1))=TFINAL(J)
80             CONTINUE
                ENDIF

C              Outlet flow rate
                OUT(2)=FLOW

                RETURN 1
                END

C*****
C
C THIS FUNCTION CALCULATES THE FIN EFFICIENCY (EFFECTIVENESS)
C OF AN ANNULAR FIN OF CONSTANT THICKNESS.
C
C ALPHA = RADIUS AT FIN BASE / RADIUS AT FIN TIP
C BETA  = RADIUS AT FIN TIP *
C       (SQRT (2 * CONVECTION COEFFICIENT /
C           FIN CONDUCTIVITY * FIN THICKNESS))
C
FUNCTION FINEFF(ALPHA,BETA)
    REAL I0,I1,K0,K1
    ALPBET = ALPHA * BETA
    CALL BESSEL2(ALPBET,I0,I1,K0,K1)

```

```

XIO = I0
XI1 = I1
XK0 = K0
XK1 = K1
CALL BESSEL2(BETA,I0,I1,K0,K1)
YI0 = I0
YI1 = I1
YK0 = K0
YK1 = K1
FINEFF = 2.*ALPHA/BETA/(1. - ALPHA**2)*(XK1*YI1 - XI1*YK1)/
      (XK0*YI1 + XIO*YK1)
RETURN
END

```

C

C*****

C*****

C

C THIS SUBROUTINE USES POLYNOMIAL APPROXIMATIONS TO EVALUATE
C THE BESSEL FUNCTIONS. THE APPROXIMATIONS ARE FROM ABRAMOWITZ
C AND STEGUN, HANDBOOK OF MATHEMATICAL FUNCTIONS, DOVER
C PUBLICATIONS, INC., NEW YORK, NY.

C

```

SUBROUTINE BESSEL2(X,I0,I1,K0,K1)
COMMON /LUNITS/ LUR,LUW,IFORM,LUK
REAL X,I0,I1,K0,K1,IT

```

C

C THE FOLLOWING DATA STATEMENTS CONTAIN THE COEFFICIENTS TO
C THE POLYNOMIALS.

C

C I0

```

DATA A0/1.0/,A1/3.5156229/,A2/3.0899424/,A3/1.2067492/
DATA A4/0.2659732/,A5/0.0360768/,A6/0.0045813/

```

C I0

```

DATA B0/0.39894228/,B1/0.01328592/,B2/0.00225319/
DATA B3/-0.00157565/,B4/0.00916281/,B5/-0.02057706/
DATA B6/0.02635537/,B7/-0.01647633/,B8/0.00392377/

```

C I1

```

DATA C0/0.5/,C1/0.87890594/,C2/0.51498869/,C3/0.15084934/
DATA C4/0.02658733/,C5/0.00301532/,C6/0.00032411/

```

C I1

```

DATA D0/0.39894228/,D1/-0.03988024/,D2/-0.00362018/
DATA D3/0.00163801/,D4/-0.01031555/,D5/0.02282967/

```

```

DATA D6/-0.02895312/,D7/0.01787654/,D8/-0.00420059/
C K0
DATA E0/-0.57721566/,E1/0.4227842/,E2/0.23069756/
DATA E3/0.0348859/,E4/0.00262698/,E5/0.0001075/,E6/0.0000074/
C K0
DATA F0/1.25331414/,F1/-0.07832358/,F2/0.02189568/
DATA F3/-0.01062446/,F4/0.00587872/,F5/-0.0025154/
DATA F6/0.00053208/
C K1
DATA G0/1.0/,G1/0.15443144/,G2/-0.67278579/,G3/-0.18156897/
DATA G4/-0.01919402/,G5/-0.00110404/,G6/-0.00004686/
C K1
DATA H0/1.25331414/,H1/0.23498619/,H2/-0.0365562/
DATA H3/0.01504268/,H4/-0.00780353/,H5/0.00325614/
DATA H6/-0.00068245/
C
IF (X .LT. -3.75) THEN
  WRITE(LUW,100) 164,52,52,X
  CALL MYSTOP(164)
  RETURN
END IF
T=X/3.75
TT=T*T
C
C I0
C
IF (X .LE. 3.75) THEN
  I0=A0+TT*(A1+TT*(A2+TT*(A3+TT*(A4+TT*(A5+TT*A6))))))
ELSE
  IT=1/T
  I0=(B0+IT*(B1+IT*(B2+IT*(B3+IT*(B4+IT*(B5+IT*(B6+IT*
. (B7+IT*B8)))))))))/(SQRT(X)*EXP(-X))
END IF
C
C I1
C
IF (X .LE. 3.75) THEN
  I1=(C0+TT*(C1+TT*(C2+TT*(C3+TT*(C4+TT*(C5+TT*C6))))))*X
ELSE
  IT=1/T
  I1=(D0+IT*(D1+IT*(D2+IT*(D3+IT*(D4+IT*(D5+IT*(D6+IT*
. (D7+IT*D8)))))))))/(SQRT(X)*EXP(-X))

```

```

      END IF
C
C K0
C
      IF (X .LE. 0.0) THEN
          WRITE(LUW,100) X
          CALL MYSTOP(1001)
          RETURN
      END IF
      X1 = (X/2.)**2
      X2 = 2./X
      IF (X .LE. 2.0) THEN
          K0=-ALOG(X/2)*I0+E0+X1*(E1+X1*(E2+X1*(E3+X1*(E4+X1*
.      (E5+X1*E6))))))
      ELSE
          K0=(F0+X2*(F1+X2*(F2+X2*(F3+X2*(F4+X2*(F5+X2*F6))))))
.      /(SQRT(X)*EXP(X))
      END IF
C
C K1
C
      IF (X .LE. 2.0) THEN
          K1=(X*ALOG(X/2.)*I1+G0+X1*(G1+X1*(G2+X1*(G3+X1*(G4+X1*
.      (G5+X1*G6)))))))/X
      ELSE
          K1=(H0+X2*(H1+X2*(H2+X2*(H3+X2*(H4+X2*(H5+X2*H6))))))
.      /(SQRT(X)*EXP(X))
      END IF
      RETURN
C
C FORMATS
C
100  FORMAT(//,1X,'***** ERROR *****',8X,'TRNSYS ERROR # ',I3,/1X,
.      'UNIT ',I3,' TYPE ',I3,' COOLING COIL',/1X,
.      'THE BESSEL FUNCTION CALLED FROM THE COOLING COIL SUBROUTINE COULD
.      NOT BE'/1X,'EVALUATED AT THE GIVEN VALUE OF ',F5.2,')
      END
C
C-----

```

C Function that calculates the heat lost from the fins

REAL FUNCTION QTHFIN(QGUESS)

```

COMMON/FINPARAMS/ L, NFINS, AFINS, DTUBE, TFINS, KFINS,
1      TIN, TAMB, FLOW, WS, TF, NODES
C      Redeclare the common variables
      REAL L, NFINS, AFINS, DTUBE, TFINS, KFINS, QTHSUM
      REAL TIN, TAMB, FLOW, WS, TF, NODES

C      Nusselt number, forced convection in air
      REAL NUF
C      Nusselt number, natural convection in air
      REAL NUN
C      Reynolds number, forced convection in air
      REAL REF
C      Equivalent diameter of the fins
      REAL DFIN
C      Temperature at the base of the fins
      REAL TBASE
C      Rayleigh number
      REAL RA
C      Maximum of forced and natural convection nusselt numbers
      REAL NU
C      Nusselt number for the fluid in the pipe
      REAL NUFLUID
C      Coefficient of convection between fin and ambient air
      REAL HCFA
C      Reynolds number for fluid
      REAL REFLUID
C      Prandtl number for fluid
      REAL PRFLUID
C      Viscosity for water
      REAL MU_W
C      Conductivity of water
      REAL K_W
C      Specific heat, water
      REAL CP_W
C      Coefficient of convection between the fin base and the water
      REAL HCFF
C      Variables used in calculation
      REAL A,B, Q1, Q2

```

```

C      Note: these are in SI units from the back of Holman
      DATA PRAIR/0.708/, CONDAIR/0.02624/,PI/3.14159/
      DATA VISAIR/15.69E-6/, ALPHA/0.22160E-4/

C      Reynolds number and prandtl number for the fluid in the pipe

      MU_W=VISCOSITY2(TF)
      K_W=CONDUCTIVITY2(TF)
      CP_W=CPWATER2(TF)

      REFLUID = 4.*FLOW/(3600.*PI*DTUBE*MU_W)
      PRFLUID = CP_W*MU_W/K_W

C      Nusselt number for the fluid in the pipe

      IF (REFLUID.GE.2300) THEN      ! then turbulent

          NUFLUID = 0.023*REFLUID**0.8*PRFLUID**0.4
      ELSE                          ! else laminar
          NUFLUID = 3.66 + (0.0668*(DTUBE/L)*REFLUID*PRFLUID)/
1          (1+0.04*((DTUBE/L)*REFLUID*PRFLUID)**0.666)
      ENDIF

C      Calculate convection coefficient between finbase and fluid

      HCFF = NUFLUID * K_W / DTUBE *3.6

C      Area of the tube inner surface

      AREAT = L/NODES * DTUBE*PI

C      Calculate the temperature at the base of the fin by assuming a
C      guessed energy transfer

      TBASE = TF - QGUESS/(HCFF*AREAT)

C      Equivalent diameter for fins

      DFIN = (4.*AFINS/PI)**0.5

C      Nusselt for forced convection of air

```

$$\text{REF} = \text{WS} * \text{DFIN} / 2 / \text{VISAIR}$$

$$\text{NUF} = 0.332 * \text{REF}^{0.5} * \text{PRAIR}^{0.33}$$

C Nusselt for natural convection of air (use TF instead of TBASE)

$$\text{RA} = 9.81 * (1 / \text{TAMB}) * \text{ABS}(\text{TF} - \text{TAMB}) * (\text{DFIN} / 2)^3 / (\text{VISAIR} * \text{ALPHA})$$

$$\text{NUN} = 0.68 + 0.670 * \text{RA}^{0.25} / (1 + (0.492 / \text{PRAIR})^{9/16})^{4/9}$$

C Take whichever nusselt number is largest

$$\text{NU} = \text{MAX}(\text{NUN}, \text{NUF})$$

C Calculate convection coefficient between fin and ambient air

$$\text{HCFA} = \text{NU} * \text{CONDAIR} / (\text{DFIN} / 2) * 3.6$$

C Calculate the fin efficiency

C $\text{ALPHA} = \text{RADIUS AT FIN BASE} / \text{RADIUS AT FIN TIP}$

C $\text{BETA} = \text{RADIUS AT FIN TIP} *$

C $(\text{SQRT}(2 * \text{CONVECTION COEFFICIENT} /$

C $\text{FIN CONDUCTIVITY} / \text{FIN THICKNESS}))$

$$\text{A} = \text{DTUBE} / \text{DFIN}$$

$$\text{B} = \text{DFIN} / 2 * (\text{SQRT}(2 * \text{HCFA} / \text{KFINS} / \text{TFINS}))$$

$$\text{FEFF} = \text{FINEFF}(\text{A}, \text{B})$$

C Calculate the area of the fin (x2 to include both sides)

$$\text{AREAF} = \text{NFINS} * \text{L} / \text{NODES} * \text{AFINS} * 2$$

C Find the value of Q using the base temp and calculate error

$$\text{Q1} = \text{FEFF} * \text{HCFA} * (\text{TBASE} - \text{TAMB}) * \text{AREAF}$$

$$\text{QTHFIN} = \text{Q1} - \text{QGUESS}$$

END

c *****

c WATER PROPERTIES, taken from the Type 60 tank model

Real FUNCTION CPWATER2(T)

real T

CPWATER2=45359-491.6*T+2.2005*T*T-.0043807*T**3+

& 3.276d-6*T**4

end

Real FUNCTION CONDUCTIVITY2(T)

real T

CONDUCTIVITY2=-.48064+.0058471*T-7.3317d-6*T*T

end

REAL FUNCTION VISCOSITY2(T)

real T

VISCOSITY2 =0.23873 - 0.26422e-02*T + 1.1062e-05*T**2

& -2.0705e-08*T**3 + 1.4593e-11*T**4

END

C-----

C Bisection method

RECURSIVE SUBROUTINE BISECTION3(FUNC,X1,X2,XACC,XROOT)

C Using bisection, finds the root of a function FUNC known to lie between

C X1 and X2. The root, returned as XROOT, will be refined until its accuracy

C is +/- XACC

COMMON/LUNITS/LUR,LUW,IFORM,LUK

REAL FMID,F,XROOT,DX,XMID

PARAMETER (JMAX=50)

FMID=FUNC(X2)

F=FUNC(X1)

IF(F*FMID.GE.0.) THEN

WRITE(LUW,'(A,I3,A,I3,A)') ERROR at Type 262 (Unit',unit,

1 ') Bisection method out of range'

CALL MYSTOP(1001)

ENDIF

```

IF(F.LT.0.) THEN
    XROOT=X1
    DX=X2-X1
ELSE
    XROOT=X2
    DX=X1-X2
ENDIF

DO 15000 J=1,JMAX
    DX=DX*.5
    XMID=XROOT+DX
    FMID=FUNC(XMID)
    IF(FMID.LT.0.) XROOT=XMID
    IF(ABS(DX).LT.XACC .OR. FMID.EQ.0.) RETURN
15000 CONTINUE
END

C-----
SUBROUTINE ZBRAC3 (FUNC,X1,X2,SUCCESS)

C    Given a function FUNC and an initial guessed range X1 to X2, the routine
C    expands the range geometrically until a root is bracketed by the returned
C    values of X1 and X2.

PARAMETER (FACTOR=1.6,NTRY=50)
LOGICAL SUCCESS
IF(X1.EQ.X2)PAUSE 'You have to guess an initial range'
F1=FUNC(X1)
F2=FUNC(X2)
SUCCESS =.TRUE.
DO 16000 J=1,NTRY
    IF (F1*F2.LT.0.)RETURN
    IF(ABS(F1).LT.ABS(F2))THEN
        X1=X1+FACTOR*(X1-X2)
        F1=FUNC(X1)
    ELSE
        X2=X2+FACTOR*(X2-X1)
        F2=FUNC(X2)
    END IF
16000 CONTINUE
SUCCESS=.FALSE.
RETURN
END

```

A4 Fortran code for the controller component

```

SUBROUTINE TYPE275 (TIME,XIN,OUT,T,DTDT,PAR,INFO,ICNTRL,*)
C*****
C Object: Bruce Hall Controller
C IISiBat Model: Bruce Hall controller
C
C Author: Joe Coventry
C Editor:
C Date: 2/9/2002 last modified: 25/5/2002
C
C Controller for Bruce Hall project. Note that this component is also useful
C for CHAPS systems with a single pump. The second 'tank pump' can simply be
C ignored.

C          STANDARD TRNSYS DECLARATIONS
              DOUBLE PRECISION XIN,OUT
              INTEGER NI,NP,ND,NO
              PARAMETER (NI=6,NP=9,NO=5,ND=0)
              INTEGER*4 INFO,ICNTRL
              REAL T,DTDT,PAR,TIME
              DIMENSION XIN(NI),OUT(NO),PAR(NP),INFO(15)
              CHARACTER*3 YCHECK(NI),OCHECK(NO)

C          PARAMETERS

C          Controller mode 1: Delta T. 2: Fixed outlet temp (not yet available)
              INTEGER ControllerMode

C          Bypass setpoint for tank temperature for diverting flow to the cooling fins
              REAL TTset

C          Upper deadband for the collector pump (pump 1)
              REAL DBupperC

C          Lower deadband for the collector pump (pump 1)
              REAL DBlowerC

C          Upper deadband for the tank pump (pump 2)
              REAL DBupperT

C          Lower deadband for the tank pump (pump 2)
              REAL DBlowerT

C          Upper deadband for the bypass valve operation.
              REAL DBupperV

C          Lower deadband for the bypass valve operation

```

```

REAL DBlowerV
C      Temperature difference between ambient and the control setpoint for
C      operation of the pumps when the valve is bypassing fluid to the cooling fins
REAL TVDT

C      INPUTS
C      Temperature at the outlet of the collectors
REAL TCout
C      Monitoring temperature for the tank
REAL TTout
C      Control signal for pump 1 at the previous iteration
INTEGER CFpump1old
C      Control signal for pump 2 at the previous iteration
INTEGER CFpump2old
C      Control signal for the bypass valve at the previous iteration
INTEGER CFvalveold
C      Ambient temperature
REAL Tamb

C      OUTPUTS

C      Control signal for pump 1
INTEGER CFpump1
C      Control signal for pump 2
INTEGER CFpump2
C      Control signal for the bypass valve
INTEGER CFvalve

C      OTHER VARIABLES

C      Setpoint for control of the bypass valve
REAL TVSet

C-----

C      IF ITS THE FIRST CALL TO THIS UNIT, DO SOME BOOKKEEPING
      IF (INFO(7).GE.0) GO TO 100

C      FIRST CALL OF SIMULATION, CALL THE TYPECK SUBROUTINE TO CHECK THAT THE
C      USER HAS PROVIDED THE CORRECT NUMBER OF INPUTS,PARAMETERS, AND DERIVS
      INFO(6)=NO
      INFO(9)=1

```

```
CALL TYPECK(1,INFO,NI,NP,ND)
```

```
RETURN 1
```

```
C          END OF THE FIRST ITERATION BOOKKEEPING
```

```
C-----
```

```
C          GET THE VALUES OF THE PARAMETERS FOR THIS COMPONENT
```

```
100    CONTINUE
```

```
C          Mode 2 not enabled
          ControllerMode=INT(PAR(1)+0.01)
          TTset=PAR(2)
          DBupperC=PAR(3)
          DBlowerC=PAR(4)
          DBupperV=PAR(5)
          DBlowerV=PAR(6)
          DBupperT=PAR(7)
          DBlowerT=PAR(8)
          TVDT=PAR(9)
```

```
C          GET THE VALUES OF THE INPUTS TO THIS COMPONENT
```

```
          TCout=XIN(1)
          TTout=XIN(2)
          CFpump1old=INT(XIN(3)+0.01)
          CFvalveold=INT(XIN(4)+0.01)
          CFpump2old=INT(XIN(5)+0.01)
          Tamb = XIN(6)
```

```
C-----
```

```
C          Set the control temp for the bypass valve
          TVset = Tamb + TVDT
```

```
C          Put in something to stop the controller sticking
```

```
          IF (INFO(7).EQ.0) THEN
              OUT(4)=0
              OUT(5)=0
          END IF
```

```
C          Main logic
```

```
C          Mode 1 - Delta T
```

```
          IF (ControllerMode.EQ.1) THEN
```

```
C          Bypass solenoid valve control
```

```

IF ((CFvalveold.EQ.1).AND.(TTout-TTset).GE.DBlowerV) THEN
    CFvalve=1
ELSEIF ((CFvalveold.EQ.1).AND.(TTout-TTset).LT.DBlowerV) THEN
    CFvalve=0
ELSEIF ((CFvalveold.EQ.0).AND.(TTout-TTset).GE.DBupperV) THEN
    CFvalve=1
ELSEIF ((CFvalveold.EQ.0).AND.(TTout-TTset).LT.DBupperV) THEN
    CFvalve=0
ENDIF

```

C Case 1 - If valve is off
IF (CFvalve.EQ.0) THEN

```

IF ((CFpump2old.EQ.1).AND.((TCout-TTout).GE.DBlowerT)) THEN
    CFpump2=1
    CFpump1=1
ELSEIF((CFpump2old.EQ.1).AND.((TCout-TTout).LT.DBlowerT))THEN
    CFpump2=0
    CFpump1=0
ELSEIF((CFpump2old.EQ.0).AND.((TCout-TTout).GE.DBupperT))THEN
    CFpump2=1
    CFpump1=1
ELSEIF((CFpump2old.EQ.0).AND.((TCout-TTout).LT.DBupperT))THEN
    CFpump2=0
    CFpump1=0
ENDIF

```

C Case 2 - when the control valve is on
ELSEIF (CFvalve.EQ.1) THEN

C Pump2 is always off
CFpump2=0

C Collector pump control

```

IF ((CFpump1old.EQ.1).AND.(TCout-TVset).GE.DBlowerC) THEN
    CFpump1=1
ELSEIF ((CFpump1old.EQ.1).AND.(TCout-TVset).LT.DBlowerC) THEN
    CFpump1=0
ELSEIF ((CFpump1old.EQ.0).AND.(TCout-TVset).GE.DBupperC) THEN
    CFpump1=1
ELSEIF ((CFpump1old.EQ.0).AND.(TCout-TVset).LT.DBupperC) THEN
    CFpump1=0

```

```

                ENDIF
            ENDIF
        ENDIF

C           Has pump1 status changed?
            IF (CFpump1.EQ.CFpump1old) THEN
C           No
                OUT(4) = INT(OUT(4)+0.1)
            ELSE
C           Yes
                OUT(4) = INT(OUT(4) +1.1)
            END IF

C           Has pump 2 status changed?
            IF (CFpump2.EQ.CFpump2old) THEN
C           No
                OUT(5) = INT(OUT(5)+0.1)
            ELSE
C           Yes
                OUT(5) = INT(OUT(5) +1.1)
            END IF

C           If this component is called 5 times or more in a timestep, then
C           simply stick to a value
            IF (OUT(4).GE.5) THEN
                CFpump1=1
            ENDIF

            IF (OUT(5).GE.5) THEN
                CFpump2=1
            ENDIF

C           SET THE OUTPUTS
200        CONTINUE
C           Collector Pump output control function
            OUT(1)=CFpump1
C           Bypass valve output control function
            OUT(2)=CFvalve
C           Tank pump output control function
            OUT(3)=CFpump2
        RETURN 1
    END

```

A5 TRNSYS deck file for the system base case

VERSION 15

*** TRNSYS input file (deck) generated by IISiBat 3

*** on Wednesday, May 26, 2004 at 13:24

*** If you edit this file, use the File/Import TRNSYS Input File function in

*** IISiBat 3 to update the project.

*** If you have problems, questions or suggestions please contact your local

*** TRNSYS distributor or <mailto:iisibat@cstb.fr>

ASSIGN C:\trnsys15\IISiBat3\Data\Joe\modeld.LST 6

*** Control cards

* START, STOP and STEP

CONSTANTS 3

START=1

STOP=8760

STEP=.1

*SIMULATION Start time End time Time step

SIMULATION START STOP STEP

* User defined CONSTANTS

*

Integration Convergence

TOLERANCES 0.001 0.001

*

Max iterations Max warnings Trace limit

LIMITS 25 999 25

*

TRNSYS numerical integration solver method

DFQ 1

*

TRNSYS output file width, number of characters

WIDTH 80

*

NOLIST statement

LIST

*

MAP statement

MAP

*

Solver statement

SOLVER 0


```

*****
*** Units
*****

* EQUATIONS "Calc. modified load"
*
EQUATIONS 1
Modload = 10*[7,1]

*-----
* EQUATIONS "Calc. HW flow"
EQUATIONS 1
mdot = Modload/(4.18*(45-[7,2]+eq1(45,[7,2])))
*-----
* EQUATIONS "Calc. solar input"
EQUATIONS 1
Qsun = 37.5*[6,8]
*-----
* EQUATIONS "Otemp calc."
EQUATIONS 1
Idbout = [6,8]*(1-[17,1])
*-----
* Model "Canberra weather" (Type 9)
UNIT 5 TYPE 9  Canberra weather

PARAMETERS 36
* 1 Mode
-1
* 2 Header Lines to Skip
0
* 3 No. of values to read
10
* 4 Time interval of data
1
* 5 Interpolate or not?-1
-1
* 6 Multiplication factor-1
1.0
* 7 Addition factor-1
0
* 8 Interpolate or not?-2
-1

```

* 9 Multiplication factor-2
1.0
* 10 Addition factor-2
0
* 11 Interpolate or not?-3
-3
* 12 Multiplication factor-3
1.0
* 13 Addition factor-3
0
* 14 Interpolate or not?-4
-4
* 15 Multiplication factor-4
10
* 16 Addition factor-4
0
* 17 Interpolate or not?-5
-5
* 18 Multiplication factor-5
10
* 19 Addition factor-5
0
* 20 Interpolate or not?-6
6
* 21 Multiplication factor-6
0.1
* 22 Addition factor-6
0
* 23 Interpolate or not?-7
7
* 24 Multiplication factor-7
0.1
* 25 Addition factor-7
0
* 26 Interpolate or not?-8
8
* 27 Multiplication factor-8
0.1
* 28 Addition factor-8
0
* 29 Interpolate or not?-9
-9

* 30 Multiplication factor-9

1.0

* 31 Addition factor-9

0

* 32 Interpolate or not?-10

-10

* 33 Multiplication factor-10

1.0

* 34 Addition factor-10

0

* 35 Logical unit

14

* 36 Format specification

1

(1x,3F2.0,5F3.0,F2.0,F1.0)

*** External files

ASSIGN C:\trnsys15\Weather\canberra.tmy 14

*-----

* Model "Tracking" (Type 16)

UNIT 6 TYPE 16 Tracking

PARAMETERS 9

* 1 Horiz. radiation mode

4

* 2 Tracking mode

3

* 3 Tilted surface mode

2

* 4 Starting day

1

* 5 Latitude

-35.2

* 6 Solar constant

4871

* 7 Shift in solar time

0.8

* 8 Not used

2

* 9 Solar time?

-1

INPUTS 7

* Canberra weather:Output 4 ->Total radiation on horizontal surface
5,4
* Canberra weather:Output 5 ->Direct normal beam radiation on horizontal
5,5
* Canberra weather:Time of last read ->Time of last data read
5,99
* Canberra weather:Time of next read ->Time of next data read
5,100
* [unconnected] Ground reflectance
0,0
* [unconnected] Slope of surface
0,0
* [unconnected] Azimuth of surface
0,0
*** INITIAL INPUT VALUES
0 0 0.0 1 0.2 19.18
36.03
*-----

* Model "AS4234 load data" (Type 9)
UNIT 7 TYPE 9 AS4234 load data

PARAMETERS 12

* 1 Mode
-1
* 2 Header Lines to Skip
2
* 3 No. of values to read
2
* 4 Time interval of data
1.0
* 5 Interpolate or not?-1
-1
* 6 Multiplication factor-1
1.0
* 7 Addition factor-1
0
* 8 Interpolate or not?-2
-2
* 9 Multiplication factor-2
1.0
* 10 Addition factor-2

0
* 11 Logical unit
11
* 12 Not used
-1
*** External files
ASSIGN C:\trnsys15\IISI\Bat3\Data\Joe\Canload.txt 11
*-----

* Model "Tempering valve" (Type 11)
UNIT 8 TYPE 11 Tempering valve

PARAMETERS 2
* 1 Tempering valve mode
4
* 2 # of oscillations allowed
7

INPUTS 4
* AS4234 load data:Output 2 ->Inlet temperature
7,2
* Calc. HW flow:mdot ->Inlet flow rate
mdot
* Tank:Temperature to load ->Heat source temperature
13,3
* [unconnected] Set point temperature
0,0

*** INITIAL INPUT VALUES
20.0 100.0 55.0 45
*-----

* Model "Pump" (Type 3)
UNIT 9 TYPE 3 Pump

PARAMETERS 5
* 1 Maximum flow rate
1100
* 2 Fluid specific heat
4.190
* 3 Maximum power
540
* 4 Conversion coefficient
0.10

* 5 Power coefficient

0.5

INPUTS 3

* T-piece:Outlet temperature ->Inlet fluid temperature

15,1

* T-piece:Outlet flow rate ->Inlet mass flow rate

15,2

* Controller:Collector Pump output control function ->Control signal

18,1

*** INITIAL INPUT VALUES

35 36 1.0

*-----

* Model "3-way valve" (Type 11)

UNIT 10 TYPE 11 3-way valve

PARAMETERS 1

* 1 Controlled flow diverter mode

2

INPUTS 3

* CHAPS collector:TOUT ->Inlet temperature

12,1

* CHAPS collector:FLOW ->Inlet flow rate

12,2

* Controller:Bypass valve output control function ->Control signal

18,2

*** INITIAL INPUT VALUES

20.0 100.0 0.5

*-----

* Model "End loss" (Type 123)

UNIT 11 TYPE 123 End loss

PARAMETERS 3

* 1 Trough Length

24.21

* 2 Trough Width

1.55

* 3 Focal Length

.845

INPUTS 1

* Tracking:Incidence angle for surface 1 ->Incidence Angle

6,10

*** INITIAL INPUT VALUES

0

*-----

* Model "CHAPS collector" (Type 262)

UNIT 12 TYPE 262 CHAPS collector

PARAMETERS 34

* 1 MODE

2

* 2 CELLS

10

* 3 REFEFF

.161

* 4 REFTEMP

65

* 5 BETA

-0.004

* 6 UNIFORMITY

.845

* 7 LENGTH

23.19

* 8 WIDTH

1.47

* 9 REFLM

0.935

* 10 SHAPE

.99

* 11 TRANSABS

0.886

* 12 COVERABS

0.063

* 13 EMIS

0.88

* 14 MASS

121.9

* 15 CP

1.08

* 16 TSTART

25

* 17 WGLASS

248

0.08

* 18 UCG

327

* 19 WCG

0.07

* 20 UCP

5787

* 21 WCP

0.04

* 22 UPT

1000000

* 23 WPT

1

* 24 UINSUL

23

* 25 WINSUL

.2

* 26 WCOVER

0.2

* 27 EMISCOV

0.1

* 28 CW0

22

* 29 CW1

27.4

* 30 CW2

-2

* 31 TOL

0.1

* 32 PERIM

0.1298

* 33 XSAREA

0.0003587

* 34 FH

0.74

INPUTS 7

* Pump:Outlet fluid temperature ->TFI

9,1

* Pump:Outlet flow rate ->FLOW

9,2

* Otemp calc.:ldbout ->ID

ldbout

* Canberra weather:Output 6 ->TAMB

5,6

* Canberra weather:Output 7 ->WIND

5,7

* End loss:End Loss Factor ->SHADE

11,1

* [unconnected] DIRT

0,0

*** INITIAL INPUT VALUES

25 1000 1000 25 2 1

1

*-----

* Model "Tank" (Type 38)

UNIT 13 TYPE 38 Tank

PARAMETERS 17

* 1 Inlet position mode

1

* 2 Tank volume

1.5

* 3 Tank height

1.53

* 4 Height of collector return

1.06

* 5 Fluid specific heat

4.190

* 6 Fluid density

1000.0

* 7 Thermal conductivity

7.2

* 8 Tank configuration

1

* 9 Overall Loss Coefficient

26.365

* 10 Insulation ratio

1.0

* 11 Initial temperature

30.0

* 12 Maximum heating rate

64800

* 13 Auxiliary height

1.2

* 14 Thermostat height

1.25

* 15 Set point temperature

65

* 16 Temperature deadband

8

* 17 Flue loss coefficient

0.0

INPUTS 6

* 3-way valve:Temperature at outlet 1 ->Hot-side temperature

10,1

* 3-way valve:Flow rate at outlet 1 ->Hot-side flowrate

10,2

* Tempering valve:Temperature at outlet 1 ->Cold-side temperature

8,1

* Tempering valve:Flowrate at outlet 1 ->Cold-side flowrate

8,2

* Canberra weather:Output 6 ->Environment temperature

5,6

* [unconnected] Control signal

0,0

*** INITIAL INPUT VALUES

45.0 100.0 20.0 100.0 22.0 1

*-----

* Model "Finned tube HE" (Type 273)

UNIT 14 TYPE 273 Finned tube HE

PARAMETERS 11

* 1 Length of heat exchanger

60

* 2 Fins per length

160

* 3 Area of each fin

0.003

* 4 Diameter of tube

0.0109

* 5 Thickness of fins

0.00025

* 6 Fin conductivity

637

* 7 NODES

10

* 8 Specific heat of fin-tube

0.4

* 9 Mass of fin tube

19

* 10 Specific heat of the fluid

4.18

* 11 Mass of fluid in the heat exchanger

60

INPUTS 4

* 3-way valve:Temperature at outlet 2 ->Temperature of inlet fluid

10,3

* 3-way valve:Flow rate at outlet 2 ->Flow rate of inlet fluid

10,4

* Canberra weather:Output 6 ->Ambient temperature

5,6

* Canberra weather:Output 7 ->Wind speed

5,7

*** INITIAL INPUT VALUES

55 625 25 1

*-----

* Model "T-piece" (Type 11)

UNIT 15 TYPE 11 T-piece

PARAMETERS 1

* 1 Tee piece mode

1

INPUTS 4

* Tank:Temperature to heat source ->Temperature at inlet 1

13,1

* Tank:Flowrate to heat source ->Flow rate at inlet 1

13,2

* Finned tube HE:Outlet fluid temperature ->Temperature at inlet 2

14,1

* Finned tube HE:Outlet flow rate ->Flow rate at inlet 2

14,2

*** INITIAL INPUT VALUES

20.0 100.0 20.0 100.0

*-----

* Model "T-piece2" (Type 11)

UNIT 16 TYPE 11 T-piece2

PARAMETERS 1

* 1 Tee piece mode

1

INPUTS 4

* Tank:Temperature to load ->Temperature at inlet 1

13,3

* Tank:Flowrate to load ->Flow rate at inlet 1

13,4

* Tempering valve:Temperature at outlet 2 ->Temperature at inlet 2

8,3

* Tempering valve:Flow rate at outlet 2 ->Flow rate at inlet 2

8,4

*** INITIAL INPUT VALUES

20.0 100.0 20.0 100.0

*-----

* Model "Otemp park" (Type 2)

UNIT 17 TYPE 2 Otemp park

PARAMETERS 2

* 1 No. of oscillations

5

* 2 High limit cut-out

200

INPUTS 6

* CHAPS collector:TOUT ->Upper input value

12,1

* [unconnected] Lower input value

0,0

* CHAPS collector:TOUT ->Monitoring value

12,1

* Otemp park:Output control function ->Input control function

17,1

* [unconnected] Upper dead band

0,0

* [unconnected] Lower dead band

0,0

*** INITIAL INPUT VALUES

20.0 200 20.0 0 5 0.5

*-----

* Model "Controller" (Type 275)

UNIT 18 TYPE 275 Controller

PARAMETERS 9

* 1 Controller mode

1

* 2 Tank upper temperature setpoint

0

* 3 Upper deadband for pump

5

* 4 Lower deadband for pump

.5

* 5 Upper deadband for bypass valve

5

* 6 Lower deadband for bypass valve

.5

* 7 Upper deadband for tank pump

5

* 8 Lower deadband for tank pump

.5

* 9 Delta T for bypass mode

5

INPUTS 6

* CHAPS collector:TOUT ->Collector outlet temperature

12,1

* Tank:Average tank temperature ->Tank monitoring temperature

13,10

* Controller:Collector Pump output control function ->collector pump input control function

18,1

* Controller:Bypass valve output control function ->Bypass valve control function

18,2

* Controller:Tank pump output control function ->Tank pump input control function

18,3

* Canberra weather:Output 6 ->Ambient temperature

5,6

*** INITIAL INPUT VALUES

0 0 1 0 1 0

*-----

* Model "Output" (Type 28)

UNIT 19 TYPE 28 Output

PARAMETERS 18

* 1 Summary interval

-1

* 2 Summary start time

1

* 3 Summary stop time

8760

* 4 Logical unit

19

* 5 Output mode

1

* 6 Operation code-1

1

* 7 Operation code-2

0

* 8 Operation code-3

-4

* 9 Operation code-4

0

* 10 Operation code-5

-4

* 11 Operation code-6

0

* 12 Operation code-7

-4

* 13 Operation code-8

0

* 14 Operation code-9

-4

* 15 Operation code-10

0

* 16 Operation code-11

-4

* 17 Operation code-12

0

* 18 Operation code-13

-4

INPUTS 6

* Tank:Internal energy change ->Summary input-1

13,7

* Tank:Energy rate to load ->Summary input-2

13,6

* Tank:Auxiliary heating rate ->Summary input-3

13,8

* CHAPS collector:QELEC ->Summary input-4

12,3

* [unconnected] Summary input-5

0,0

* Calc. solar input:Qsun ->Summary input-6

Qsun

LABELS 6

DE Qload Qaux Qelec1 Qelec2 Qsun

*** External files

ASSIGN C:\trnsys15\IISI\Bat3\Data\Joe\modeld.out 19

*-----

END

Appendix B

Appendix B contains the experimental raw data used for TRNSYS validations. The shaded sections show the periods of steady state measurement.

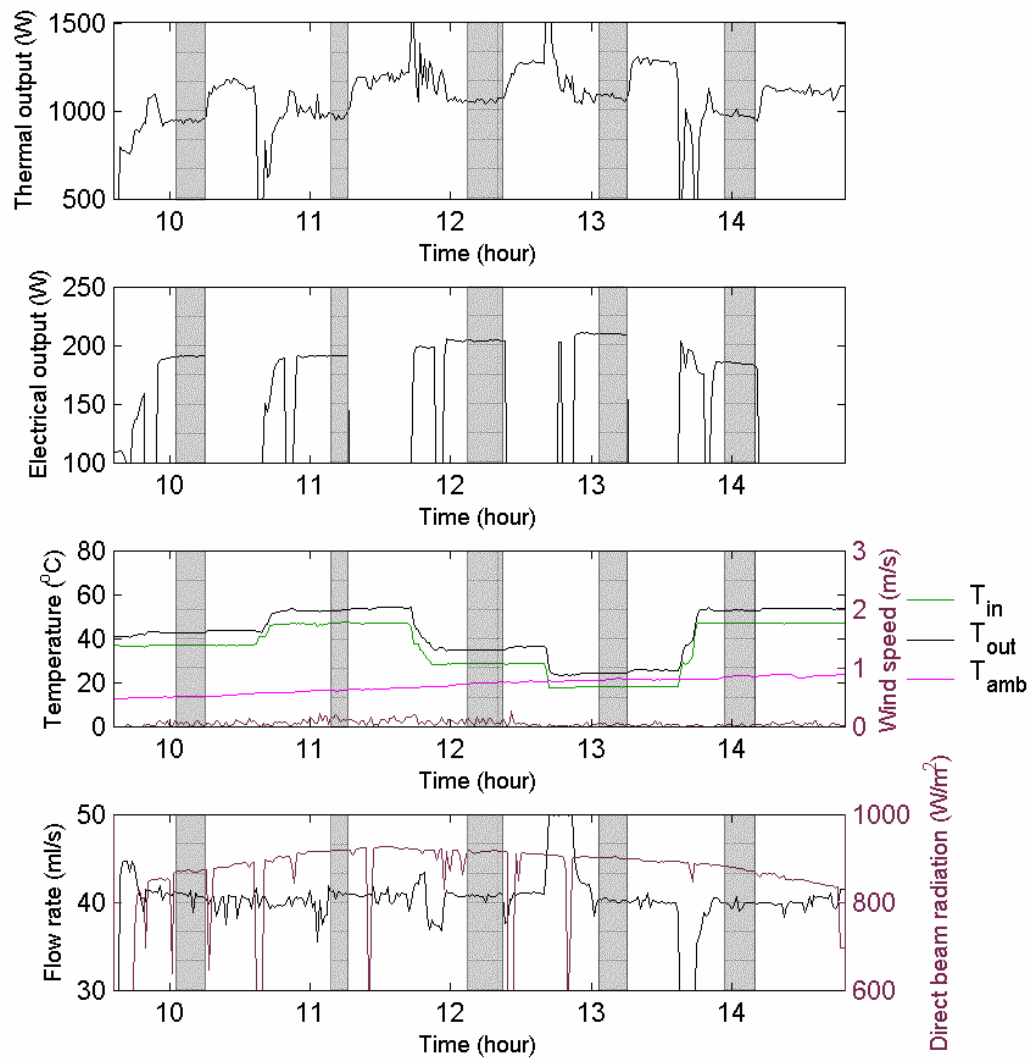


Figure B1. Measured data from 5 May 2003. The shaded areas show the steady state data used in the efficiency curves.

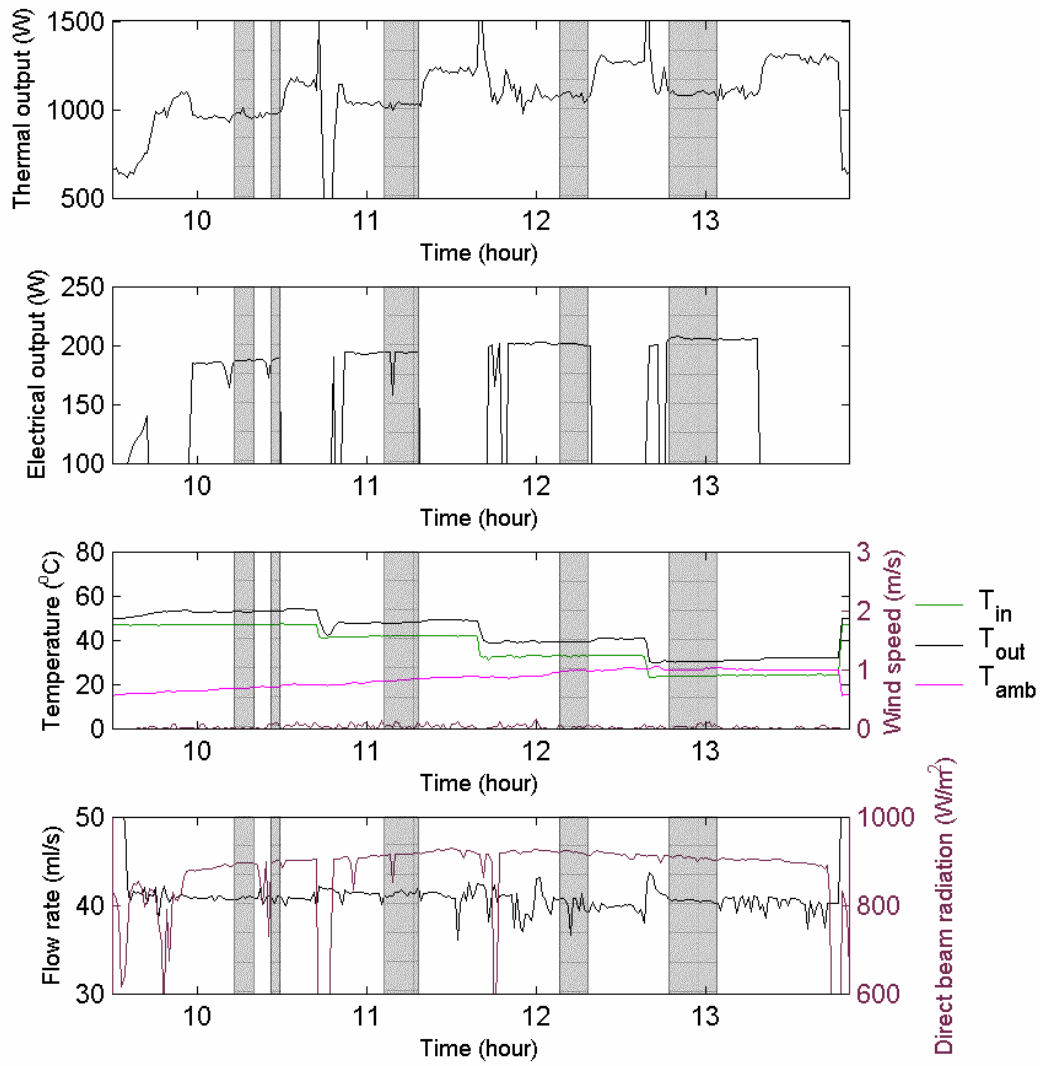


Figure B2. Measured and simulated data from 6 May 2003. The shaded areas show the steady state data used in the efficiency curves.

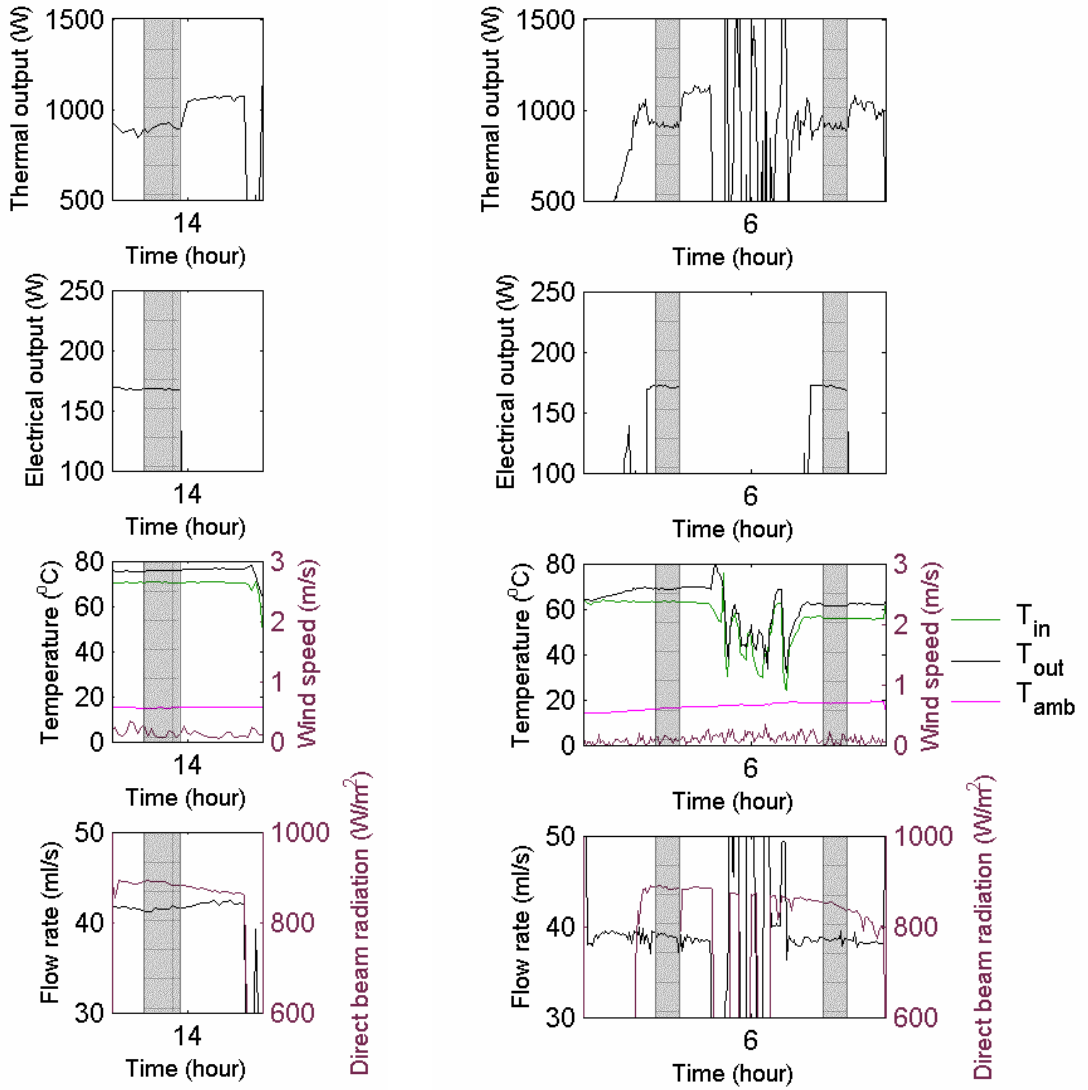


Figure B3. Measured and simulated data from 22 May 2003 (left) and the 29 May 2003 (right). The shaded areas show the steady state data used in the efficiency curves.

Appendix C

Franklin, E.T., Coventry, J.S. Effects of Highly Non-uniform Illumination Distribution on Electrical Performance of Solar Cells. *Proceedings of Solar 2002 Australian and New Zealand Solar Energy Society Paper 1*.

Copy of article available in hard copy of thesis held in Library.