

ASSEMBLY NOTES
MACHINE NO 2

MONTPELLIER



Shrink sleeve on intensifier cylinder:

Max dia of cylinder = 160.07
Min dia of sleeve = 160.04 } so 0.03 interference

Assembled OK with 200°C heating.

Intensifier oil cylinder 2302

Sent back to Sydney for thread chasing. It then had 0.3 end play.

The i.d. of the cylinder varied from 130.120 to 130.137 as ground.

At 1000 MPa, the diameter stretch is 0.178 mm, so question is whether $0.178 + 0.137 = 0.315$ mm is too much for the 2 mm nitro ring to cope with. At 700 MPa, it is $0.125 + 0.137 = 0.26$ mm - probably OK.

Bore re-measured with 3 point micrometer

One end: 130.155 end
130.130 in about 50 mm
can't reach middle
130.120 in about 50
130.135 other end

10/1/91

Top end 130.168
~ 130.130 50 mm down

Female

$$PD = D_{\text{measured}} + 3.804$$

with 3mm wire

Male

$$PD = D_{\text{measured}} - 6.8038$$

with 4mm wire

These nuts have been transferred to No 1 vessel.

New nuts made
— see later.

No 2 Cylinder

Delivered 30/3/90
Cradle damaged
Ventil seems OK

Top female thread. (Measured before shrinking jacket)
Measured between 3mm wires, dia = 152.25

→ PD = 156.05 outside limits
to 155.96
152.16 inner end.
156.103 - 156.503
for 6H fit.

Bottom female thread 152.39 — 152.43 (oval)
152.37 — 152.43

→ PD = 156.194 — 156.234
156.174 — 156.234
within limits but at
least 0.040 out of round

NUTS

Top nut ~~is~~: D_{max} = 162.08
162.08

limits
156.023 } 6g
155.723 }

→ PD = 155.276

Bottom nut: D_{max} = 162.35
162.35

→ PD = 155.546

} both outside
limits

Bore of No 2 vessel

at top 64.90

20mm in from top: 64.925

20 " " bottom: 65.075

at bottom 65.075

Cylinder Minor diameters:

Top { 153.6
153.7

limits

153.505

- 154.305

Bottom 153.86 - 153.95

153.85 - 153.78

Within limits

Conclusions: Threads in vessel a little tight at top, but within limits at bottom although out of round. Minor diam's within limits

Nuts both outside limits, too small by 0.2 to 0.5 mm at least, preferably 0.4 to 0.7 mm bigger.

Aug 17, 1990

New nuts for No 2 Vessel

Top nut : Measured over $\approx 4\phi$ wires : 162.92

→ P.D. = 156.12

Crest diam. = 159.74 top
159.66 bottom

Does not fit : PD too big ; vessel thread should be chared out to fit nut.

Bottom nut : Over wires : 162.62

→ P.D. = 155.82

Crest diam = 159.53 parallel.

Just fits — bottom thread ^{of vessel} could do with a skim to take out out-of-round & give a freer fit.
Probably should not be cleaned up altogether.

HARDNESS OF NO 2 VESSEL

(heat-treated by Gow's J, 23-2-90)
using vacuum furnace

600°C 30 min

800°C 2 hrs

1020°C 3½ hr

nitrogen quench

Temper 520°C 3½ hr

" 620 4½ hrs

Measured at 46 HRC

Powarbo order TO 101
23-2-90

Invoice R8547139

would not
have been
tempered
in vac.
furnace

Info. from Kerry Weight, manager.

Tested on bottom end by Hawker de Havilland 9-7-90:

Three positions gave 393 HB 42 HRC

4106 HB 44

420 HB 45

However, tests on 1½-2 mm thick slices cut by EDM
from about 10 mm from ends gave:

Near bottom end (no 2) : 41½, 42½, 41½ HRC

Near top end (no 1) : 50, 49, 48½ HRC

Equotip tests from Metlab Maple give 45 HRC
at one end and 51 HRC at the other ; 49 HRC
half way along.

Further tests on hardness of vessel 14 Sept 1990

Top end tested by de Havelland 10/9/90 :

Three positions :

505 HB	52 HRC
514 HB	52 HRC
514 HB	52 HRC

Also further results (certificate) on the EDM slices :

Near Bottom end : 41.5-42.5 HRC directly

42.5-44.5 " in traverse by HV_{0.2} (microhardness)

Near Top end : 48.5-50 HRC directly

50.5-53.5 " in traverse.

Thus the hardness is OK at bottom end and about 100 HB too high at the top end.

Based on tempering $T = 600^{\circ}\text{C}$ the diagram on p19 of the ASSAB Tool Steel Handbook suggests that the quenching temperature must have been somewhat over 1100°C at the top end and somewhat below 900°C at the bottom end. Gow's card says it was austemitized at 1020°C , for which the tempering diagram would predict around 47 HRC for a 600°C temper. Alternatively, if the austemitizing temp were uniform, the tempering temperature must have varied from about 550°C at the top to 630°C at the bottom (opposite sense of temp. gradient to that for hardening).

21/9/90

Discussion with Kerry Weight (foreman) and Sean Harrington (manager) at Gov's Heat Treatment suggested that the vessel may have been tempered in a smaller, muffle furnace adjacent to the vacuum furnace (often done to conserve energy unless vacuum furnace tempering is specified) and that this furnace may have been cooler nearer the door. This corroborates the view of Barry Moore (Asst) that the hardness variation is probably a tempering problem.

INTENSIFIER CYLINDER No 2

Hardness was tested by Hawker de Havilland on 21/9/90 (This cylinder was re-stamped no. 1 on the outer sleeve by me in mistake, not finding the original no 2 , and so it appears as no. 1 on H.deH.'s certificate

The pressure connection end gave

HB 432, 429, 429 , equist to HRC 46

and the threaded end gave

HB 388, 390, 390 , equist to HRC 42

While 42 ± 2 HRC was the specification, the higher hardness at the top end is acceptable in the intensifier cylinder.

PRESSURE VESSEL No 2 - RETEMPER

According to telephone call to Kevin Sheddor,
The vessel was 40 HRC one end and
43 HRC the other after Gow's had re-
tempered (they were supposed to do it
in the vacuum furnace).

Then I agreed that the vessel be re-ground
as follows:

- OD to specified size (230-100 / 230-060)
- seal bore each end - 67-150 / 67-180
- main bore 65-450 / 65-500 with
honed finish
- bottom of bore 145-150 / 145-210
- cham threads to remove distortion

Plugs (top & bottom) to be 67-140 / 67-120
145-100 / 145-060

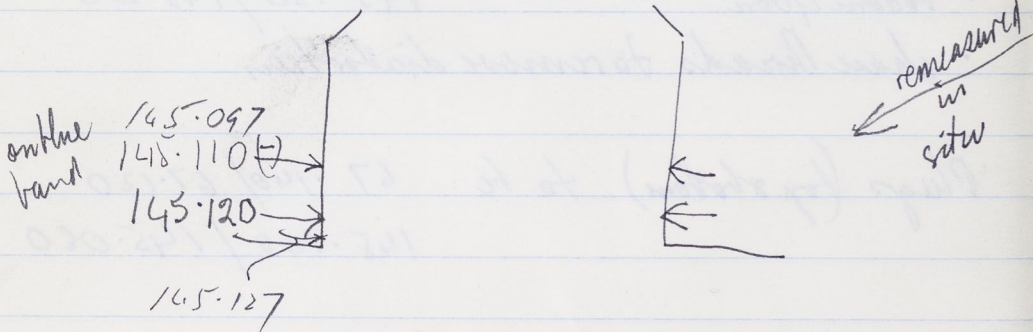
New furnace & load cell plugs were made.

ID checked in situ from top

~~65.650~~ (been honed)

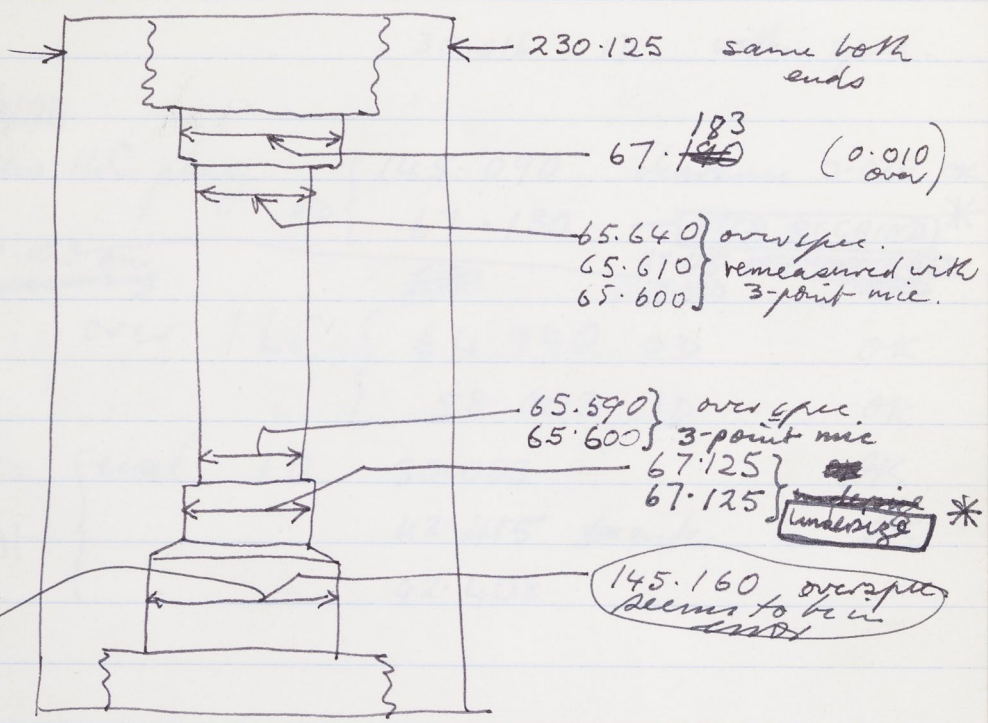
65.650 (micrometer a bit cold; may
be a few μm too big
~20 mm below O-ring

65.582 as far down as can reach
(65 below O-ring groove)



9.1.91

Pressure Vessel dimensions after re-grind:



Vessel threads been chased (a bit of chatter)
 Top nut goes in OK: been semi-blacked
 Bottom nut " " "
 Inner top nut fits OK.

Not been remachined on bottom face * — done by Frank 24/1/91

DRG 3101

New furnace plug : 67.190 O.D.

Clearance in box = 0.007 - rather small
30.016 ID within spec.

actually
- 0.007
ie interference
fit.

DRG 4101

New ILC plug : { 145.090 clearance 0.070 OK.
OD { 67.130 - **NEED REGRIND** *

67.103 on
remeasuring

~~67.~~

67.115 ~~67.110~~ ~~67.115~~
67.090

over ILC { 64.940 OD OK
58.003 ID OK.

checked
later
9/4/91

{ seal ID 30.008 OK
" " 42.415 ~~outer~~ OK
" " 42.408 "

25/1/91

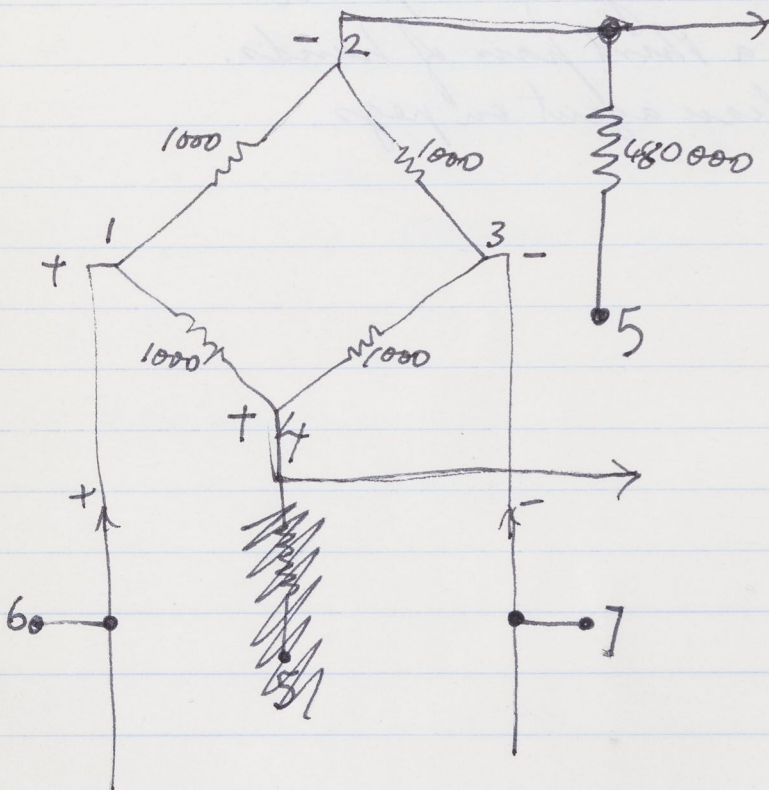
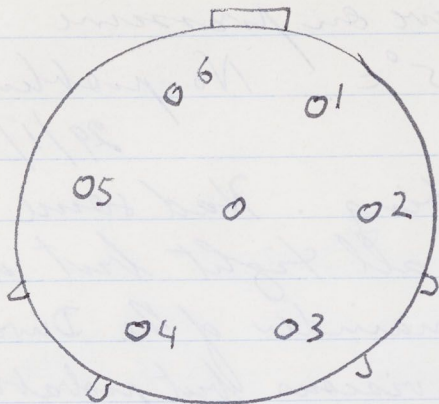
Shrank outer sleeve on pressure vessel, from 225°C. No problem.

29/1/91

Wound copper tubing. Had some difficulties getting all tight but not too bad. Used remainder of the Duwaleo 132 cement. Pretty viscous but probably necessary for staying in place.

Could use a third pair of hands.

Also put release agent on pegs.



TOP Pressure Gauge
Labelled ASCO INSTRUMENTS

PR 851 7000 bars 129

Resistances measured with Wheatstone bridge
(leads 0.08 Ω)

	Ω		Ω
1 - 3	1004	2 - 3	753
- 2	753	- 4	1004
- 4	753	- 5	480000
- 5	470000	- 6	752
- 6	0.090	- 7	752
- 7	1004		

3 - 4	753	4 - 5	470000
- 5	470000	- 6	753
- 6	1004	- 7	753
- 7	0.080		

5 - 6	470000	6 - 7	1004
- 7	470000		

$$V_{in 0} = 0.496$$

$$V_{in 1} = 11.750$$

$$V_{out 0} = 0$$

$$V_{out 1} = 700$$

$$0.496$$

$$\frac{11.254}{0.496}$$

$$22.69$$

$$\left. \begin{array}{l} 700 \text{ MPa} = 11.254 \text{ mV} \\ 20 = 0.322 \text{ mV} \end{array} \right\} \text{to change } V_{in 0} \rightarrow \cancel{0.174} 0.818$$

$$V_{in 1} = \cancel{11.254} 12.072$$

↓
see 9.3-91

Original dia - see opp
9/9/91

6-3-91

First pressurization
Set gauge as at left.

Went up to 22.8 PV, 47 MPa on Heise.
Left over night.

Next morning 22.2 45.2

Changed excitation from 5V to 10V; PV now
reading ~64. Came back to zero = 20.0

Reset config as at left. 8/3/91

<u>PV</u>	<u>Heise</u>	Put in new valves & solenoid in valve	
10.0	10.5	Protair intensifier in & out	
41.7	44	intensifier in & out	
9.7	11.0	.	
97.0	102.0	.	
200.9	209.2	190	
303.9	315.0	300	
403.9	416.0	402	
497	513	500	intensifier leak

dropped back

340.5 350.0 330 11.34

slow leak. Released

0.1 ~ -20 D later back to +10

$ID = 65 \cdot 65 \cdot 35$ down 20mm
 $65 \cdot 508 \cdot 580$

ie no signif stroke at this stage.

Re-zeroed Heise

	PV	Heise	Protair
Zero:	0.0	0	0
Bottle	9.6	10.0	
Primed	98.8	102.7	94
	205.4	211.0	196
	310.9	319.9	307
	407.6	417.3	402
	502.0	514.0	504

Moderate leak; not in p.v. when valve off; not showing in bubbles ~~of~~ intensifier:
 $\Delta = 7.0$

413.0	404.0	413.0	395
304.4	7.3	311.7	296
206.1		211.0	204
108.7		111.0	105
55.9		55.7	48
0.3		-1.0	0
0.1		0.0	0 next morning

~~From graphical plot the Heise gauge reads $\frac{1.026}{1.058}$ times the PV gauge, ie have to increase the scale of PV by factor ~~1.058~~ $\frac{1.026}{1.058}$, ie 11.254 mV \rightarrow ~~11.908~~ $\frac{11.547}{1.026}$~~
 ie $V_{in 0} = \frac{0.496}{1.026} = 0.818$
 $V_{in 1} = \frac{12.043}{1.026} = 12.365$

reset
after
~~calib~~ test
to 730 Pa

see 16/3/91

Calibration

9-3-91

From a graphical plot of PV vs Heise, the Heise gauge reads 1.026, times the PV, i.e. have to increase PV readings by factor 1.026, so instead of FS being 11.254 mV, need to set FS = $11.254 / 1.026 = 10.968$ mV (presumably 10 V excitation is a bit low)

$\therefore V_{in} 0 = 0.818$ $V_{out} 0 = 0$
 $V_{in} 1 = 11.786$ $V_{out} 1 = 700$

Pumped up

PV	Heise	Protein	
202.8	203.2	186	1/2 hr wait
298.4	298.2	282	1 hr "
393.2	391.0	376	3/4 hr. v. slow leak
509	507	498	~ 1/2 min
502.6	501		1/2 min
498.7	497.0		2 1/2 "
393.2	391.3	382	1/2 hr
295.2	295.2	285	"
204.9	204.8	197	

11/3/91

Took off Heise gauge for higher pressure tests.
Bottle pressure 9 MPa.

Up to 540, leak; tightened outlet from
"intens. in" valve.

Up to 600 - still some leak; back to zero
Can't see where leak is.

I.D. = 65.580 down 65 mm

= 65.640 " 20 "

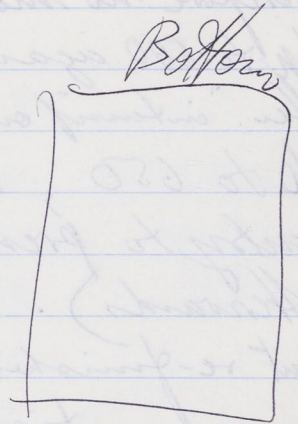
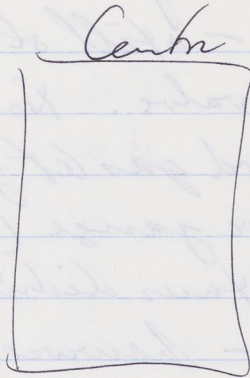
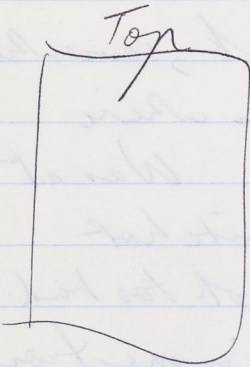
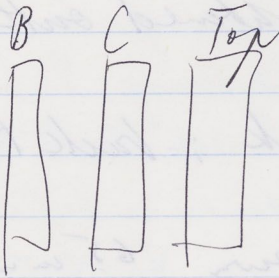
Therefore no movement yet.

Up to 600 again - still sl. leak, somewhere
after "intensify out" valve. No sign where

Up to 650, heard gas let go. Was at
entry to pressure gauge (quite hot
afterwards). Surfaces didn't look too bad
but re-finished - however on inspection
some question of a crack. So removed
top gauge, plugged & then pumped up
~~by~~ by viewing Profais oil gauge. Got to
640 bar on Profais, probably 650 MPa
but leaking as fast as pumping.

Closing "intensify out" gauge slowed up leak
rate. Plumbing leak - tightened.

Another cycle: 660 MPa. This time leak at
entrance to intensifier



13/3/91

Tightened intensifier connection.

Put gauge back in.

Up to 684, leak. — two cone seals

Up to ~ ~~713~~ 713 — cone seal

Up to 724 —

Up to 732 — holding reasonably.

ID = 65.580	down 65 mm
= 65.645	" 20 mm

ie no
measurable
stretch

Pressure zero now shifted to 2.8

14/3/91

Went to put in new (2A) furnace plug & found it to be oversize. So changed to no 3 plug (Frank had original 2 at home).

Bottle pressure: $8.4 - 2.8 = 5.6$

Pumped to ~ 80 MPa.

Tried furnace — shorted at top & bottom leads, seems to be at the brass rods. Blew top fuse. Earlier had to swap leads into fuse holders to set up top & bottom connected to correct meters.

15/3/91

Frank reorganized feed thro' with heat shrink systems to avoid shorting (most of the leads had shorted).

Pumped up again - leak at top plug ~~at~~; dropped pressure - stopped at 16

pumped - leak started at 41 again.

Repeated exactly on next cycle.

Finally spec jacket leak appeared so abandoned test.

Turned out that I had forgotten to put the inner sleeve in, so top jacket seal O-ring was beginning to be cooked. The leak was probably at the joint of PSZ & Al₂O₃ due to no chamfer in Pul PSZ.

16/3/91

Resetting zero of pressure gauge:

Previous setting was $V_{in} 0 = 0.818$ } 700 MPa
 $V_{in} 1 = 11.786$ } = 10.968 mV

which after the 730 MPa test gave a zero of 2.817 Pa
2.817 Pa equiv to $\frac{2.8}{700} \times 10.968 = 0.044$ mV, so added this to both above readings, now

being $V_{in} 0 = 0.862$ } 700 MPa $\equiv 10.968$ mV,
 $V_{out} 0 = 11.830$ }

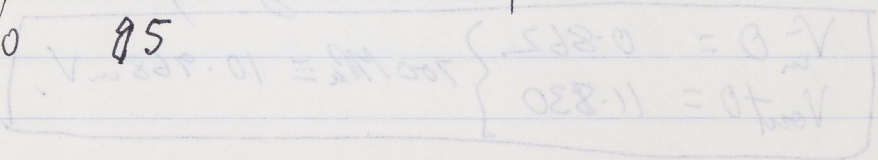
(1)

Name furnace in for machine.

(3)

Ramping 30°/min

0	243		235
1	416	(12) 529	373
2	628	649	554
3	752		656
4	857		705
5	895		721
6	924		727
7	932		729
8	952		732
9	1007		730
10	1018 - 30 / 1040		696 / 719
11	1044	1073	675
12	989		630
13	891		574
14	770		500
15	590		395
16	394		275
17	278		168
18	155		114
19	112		84
20	85		



14/3/91

Put new jacket on calib ~~#~~ pistons, after chamfering PSZ a little.

Have to be very careful of nitre ring when re-inserting — needs the O-ring compressing washer reduced a bit ~~at~~ at top jacket seal so as to reduce gap between nut & piston.

Pumped up from 4 MPa gas — a bit slow.

Profile ① after c self-time.

Control Point ⑤ = 500 0.56/0.20/1.00

Power varying too much to measure

Changed to control on ⑥, SP 550, 0.56/0.2/1.00

Started profile ② but soon abandoned % spec. leak. Pressure ~ 280 MPa

There is also still a top plug leak.

Sprang a spec. leak so abandoned.

Same problem as before — put more chamfer on PSZ piece.

Pumped up again. Changed furnace settings to 0.4/0.2/1.00 & SP to 450

Follows ramp at about 75% power (9.5, 5 & 12.5 A in TC & B resp.)

Pumped to 280 MPa (at ~300°C).

At 315 MPa, bubbles started at no 2.

No bubbles at no 3 so far.

Profile #3 329 MPa, SP6 450 0.40/0.20/1.00

	T	C	B	
Amps	8.2	3.9	12.1	}
Volts	17.4	13	12	
Watts	143	51	145	339W. 0.46 W/K
R				
Temp	472	434	448	

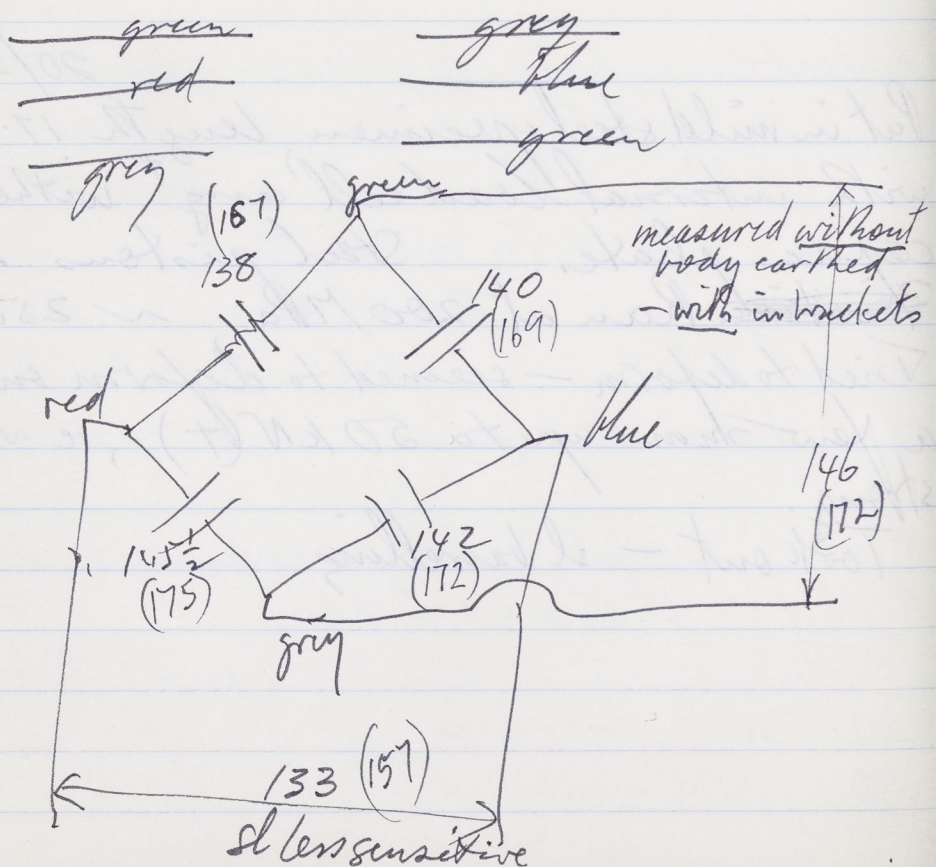
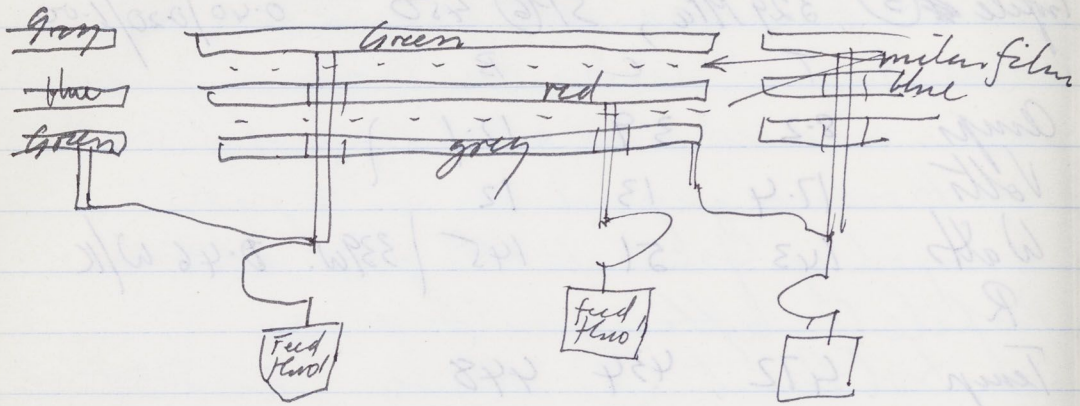
20/3/91

Put in mild steel specimen length 17.5, with internal load cell ^{ang^t} without capac. plates. Steel pistons. 26

~~Friction~~ Ran at 200 MPa ~ 250°C

Tried to deform - seemed to deform only a few mm up to 50 kN (T), i.e. very strong.

Took out - sl barrelling



Internal Load Cell

Set up similarly to that in No 1 machine except that the connections to the plates were all brought out downwards thro' holes in the lower plates & the connections made more directly to the feed throughs. An extra layer of teflon insulation (1.1 ϕ) was put over the leads down the stem, which fitted into 1mm wide grooves.

First measurements on diagram opposite.

Red plate needs to be raised a little. After readjust,

Red-green = 170

Red-grey = 170

So need to lower blue plate v. slightly. Readj to

Blue-green = 170(-)

Blue-grey = 169

Should be OK now

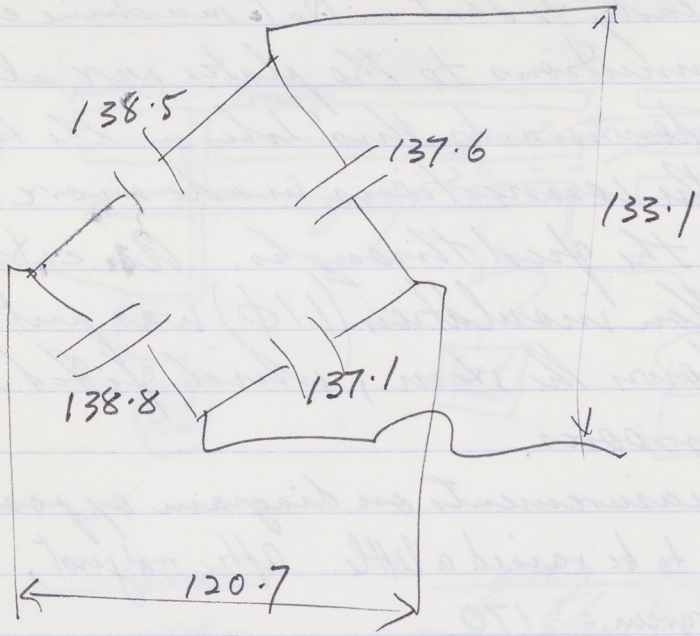
Green-grey = 170

Red-blue = 156 ← suggestive of some

sort of minor short or leakage.

Readjusted later - see over.

This time the lead from preamp to front panel was taken through a copper



H A I 1	10.0 50.0	} acts on doors
L A I 1	1.0	
H Y S 1	2.0	
H i r 1	nonlatch	
H A I 2	50.0	} acts on pumps
L A I 2	240.1	
H Y S 2	2.0	
H i r 2	nonlatch	

→ 100

conduit for more shielding, & was
terminated at the meter, to which the
power for the preamp was also brought.
Meter reading now ~~quite~~ very steady.

Capacity readings now as at left
Set zero on meter - it seems to vary a bit
from one loading to another, by about 0.1
kN; seems a bit too much to be friction
on the assembly piston. Need to check with
specimen removed.

Loaded up to 6 MPa on oil pressure
gauge (area of jack = 0.00344 m^2 ,
so 1 MPa \rightarrow 3.44 kN/MPa). Set
Calib roughly at this.

ILC zero at zero p fairly close to 0.0

On applying bottle pressure (15 MPa) zero
of ILC went up to 4.16, settled down
to 4.02, $p = 16.5$

Pumped to	55.0		ILC	11.70
	101	100		21.29
				20.93
$\frac{1}{4} \text{ hr}$	99.5			20.67
so	97.0			20.09
	96.6			20.01

100 MPa

Needs $2.8 \text{ MPa} = 9.6 \text{ kN}$ to overcome friction
at 100 MPa pressure

$$\begin{array}{r}
 35 \text{ kN reading at } 8.0 \text{ MPa} \\
 - 20 \\
 \hline
 15 \text{ kN signal}
 \end{array}
 \quad
 \begin{array}{r}
 - 2.8 \\
 \hline
 5.2 \text{ MPa} \approx 17.9, \text{ say } 18 \\
 \longrightarrow \approx 18 \text{ kN load.}
 \end{array}$$

Release oil pressure, reading = 31.8

$$\begin{array}{r}
 - 20.0 \text{ zero} \\
 \hline
 11.8 \text{ kN residual friction}
 \end{array}$$

200 MPa

Pumped up:

201 MPa

33.0

199

33.33

40°C

201

33.34

200

32.71

29°C

Friction about $5.2 \text{ MPa} = 18 \text{ kN}$

zero now ~ 31.48 , 199.5 MPa

contact 16.47 mm, pumped to 10 MPa

44.0

= 5.2 fr.

- 31.5

4.8 MPa

= 16.5 kN

12.5 kN reading

release oil pressure, reading

44.3

31.5

12.8 kN

residual friction

300 MPa near end of stroke (started about $\frac{1}{4}$ stroke)

300 MPa

301

300

301

$\frac{3}{4}$ in

298

ILC

30.85

35°

30.80

30.75

33°

30.47

29°

Friction nearly 8 MPa = 27 kN up

$6\frac{1}{2}$ MPa = 22 kN down

ILC zero movement around a bit with piston movement

pos 17.0

30.72

16.7

29.63

16.5

29.63

contact around here →

(16.35)

33.88

12 MPa

(16.23)

45.0

friction 6

-30.0

6 MPa = 20.6 kN

15.0 kN

Release pressure

45.7

30.0

15.7 kN friction

16.5

30.67

16.7

30.65

17.0

30.64

Friction consistently

17.5

30.69

~ 6 MPa = 21 kN

17.0

29.56

16.7

29.54

16.5

29.54

16.7 30.57

17.0 30.57

17.5 30.63

Seems to be a drag of ± 0.5 kN on the load cell — due to specimen assembly being eccentric??

Switched on FURNACE SP 300 ,
ramp 30 , control on T/C ⑥
T/C ⑥ = 200 P = 336 T/C ② = 63

ILC = 30.43 POS = 17.50

T/C ⑥ = 250 P = ~~349~~ T/C ② = 86 ILC = 30.30

300 364 113 30.13
overshot 5-6°C

$\frac{1}{2}$ hr: 300 390 371 29.55

Took ~ 9 MPa to move $\rightarrow 31$ kN

$T^{\frac{1}{2}}$ to keep moving $\rightarrow 26$ kN

Load zero = 28.25 moving up

44.5 at 14.0 MPa

= 48.1 kN

28

25.8 zero

16.2 kN \leftrightarrow 22.3 kN

ie 16 kN registered for 22 kN actual

Friction back = 6.8 MPa = 23 kN

T/C ② went on up to 378°C . Bottom current
12A

Was ~ 6 kN in no ① & ③ -
5.5 kN later in ②

0.030 MIN
on 42.4 ϕ

Furnace off.

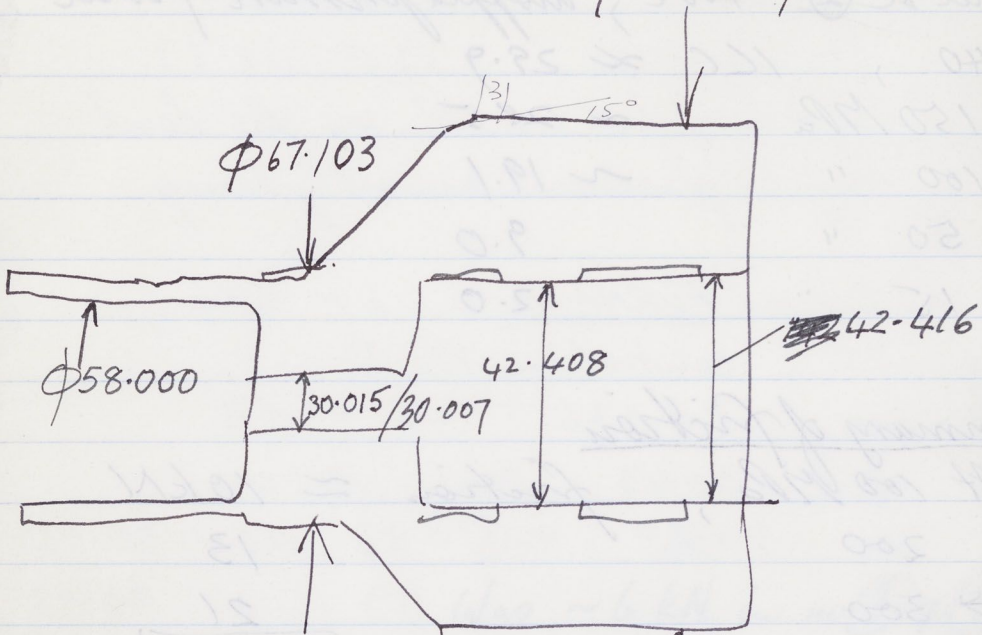
When TC (2) $\sim 100^{\circ}\text{C}$, dropped pressure from
 ~ 340 , ILC ≈ 29.9
150 MPa ~ 28.5
100 " ~ 19.1
50 " 9.0
15 " 2.0

Summary of friction

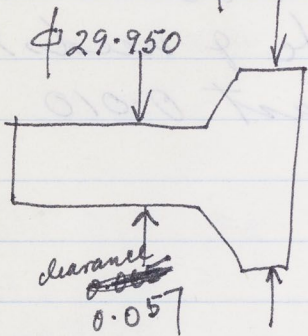
At 100 MPa, friction $\approx 10 \text{ kN}$
200 13
 \rightarrow 300 21
(see later)

So it appears that the piston clearance (0.040 mm) is inadequate & needs to be increased, by at least 0.010

$\phi 145.095 / 145.100$



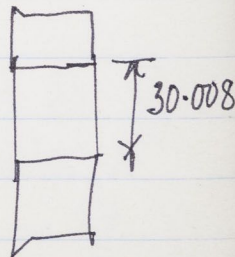
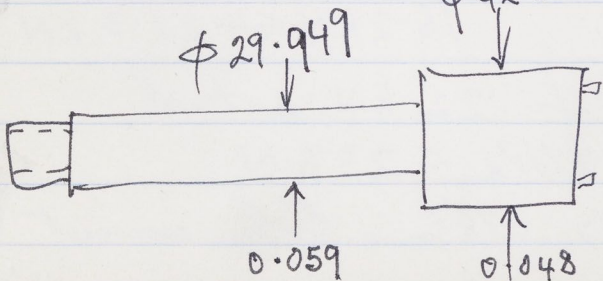
$\phi 57.990$



reduced this diam to $\phi 145.070$ to overcome difficulty of getting nut right in

$\phi 29.949$

$\phi 42.360$

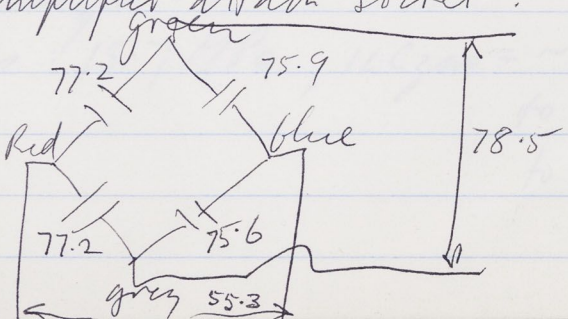


Dismantled bottom plug

Plug does not go right in — short by ~2mm. Appears that the R3 on the pressure vessel is too large, so a 15° x 3 mm taper was machined on the OD of the plug ~~4103~~ 4101 at the edge of the 45° taper.

The $\phi 58$ guide surface for the ILC piston is strongly marked as if it were a bit tight, but it is hard to imagine that this could generate 25kN of friction. However, it does appear, from extrusion of the O-ring, that the ILC is not seating properly & therefore there will be compressive force on the threads under pressure, tending to increase the OD. Relieved ILC support (4103 (12)) so that ILC seats properly. Re-assembled.

Rechecked ILC by measuring with Escort meter on pins at amplifier attach⁺ socket:



contact
19.3

11/4/91

Gas 1kC POS Pump oil pressure
52.5 MPa -3.18 20.14
 -3.21 } 18.48 MPa
 +15.27 }
 -3.18 }
 free
 Friction below gauge level.

100 MPa ~ -3.4 first 19.96
 102.2 → -4.13 } 17.33
 +13.20 }
 1.84 residual 6 MPa → 20.6 kN
 ie 5.9 kN ~~friction~~ double friction
 ~ 3 kN friction (0.9 MPa - does not register)
 zlw -4.04

200 MPa -4.96 seemed to settle down quicker
 Total Friction 2 MPa = 7 kN
 9.53 14.5 kN 6 kN - 2 = 4 = 13.7 kN
 22.5 27.5 kN 10 - 2 = 8 = 27.5
 4.47 residual
 ≈ 9.4 kN double fr = 4.7 kN friction
 zlw -4.81

There seem to be transient heavy effects
in the PLC gas after pumping

300 MPa

immediate zero = -5.33
after 5 mins -5.43
(went thro. sl higher values)
-5.37

Friction at
first ~ 5 MPa = 17
falling to ~ 3 MPa
= 10 kN

-12.9 = 18.2 kN

8 MPa
- 2.5
5.5 = 18.9 kN

-27.6 = 33.0 kN

12 MPa
- 2.5
9.5 = 32.6 kN

release pressure → 5.68

ie ~ 11 kN double friction
~ 5.5 kN friction

zero -5.26
-5.28

400 MPa

immediate zero = -5.62

5 mins -5.73

1/2 hr -5.68

-5.72

Friction down

4 MPa = 14 kN

9.4 15.1 kN 8 MPa - 4 = 13.7 kN

24.2 29.9 kN 12 - 4 = 27.5

release pressure → 8.1 dropping to 7.5 & further
= 13 kN double fr
6.5 kN friction
Pumping 3 1/2 MPa = 12 kN

zero -5.63

26.6 \equiv 32.2 kN Up to 12 MPa gauge

- 3 $\frac{1}{2}$ friction \equiv 12 kN

clean oil \rightarrow 4.8

8 $\frac{1}{2}$ \equiv 29 kN

double friction 10

friction = 5 kN

Spring a leak \sim 460 MPa

Dropped to 99 MPa

ILC zero \sim -3.8

after $\frac{1}{4}$ hr ~~102~~ 101.7 MPa

ILC zero -3.86

Released gas to zero pressure

ILC zero = -0.25

P gauge zero 0.5

\downarrow

after \sim 15 mins. -0.04

\rightarrow wandering around a bit 0.3/0.7

~~Friction up \sim 38 MPa = 13 kN~~

[Faint, mostly illegible handwritten notes and scribbles at the top of the page.]

[Faint handwritten notes in the middle section, including some numbers and symbols.]

Hold pressure
well at 6000

11/4/91

Dismantled Enerpac pump & cleaned it out for more reliable running.

Tightened up 700 MPa rupture disc.

ILC load zero -0.004 to -0.008 depending on exact pumping/loading prohistory

Pressure zero around 0.3, varying ± 0.1 MPa.

Bottle pressure 15 MPa

ELC zero -1.15

400 MPa

immediate zero -5.92

after $\sim 1/2$ hr -5.79

Friction up ~ 3.817 Pa $\equiv 13$ kN
zero 5.73

10.86

7.817 Pa $-3.8 = 4.0$

$\equiv 16.6$ kN

$\equiv 13.7$ kN

26.3

12 ~~MPa~~ $-3.8 = 8.2$ MPa

+ 5.7

$\equiv 32$ kN

$\equiv 28$ kN

Released pressure $\rightarrow \sim 4.5$ kN

$= 10$ kN ^{double} friction, dropping back

$= 5$ kN friction

Pumped back

zero $= -5.69, -5.70$

Intensifier occasionally springing
a peak near 500; dropping
back, came up OK.

500 MPa

zero soon after reaching p : -5.06

after few mins -6.01

after dropping back in oil -5.96

Friction up $\sim 4.2 \text{ MPa} = 14.4 \text{ kN}$

zero -5.92

15.3

10.0 MPa

+5.92

-4.2

= 21.2 kN

$\frac{5.8 \text{ MPa}}{0.29} = 20 \text{ MPa}$

Released pressure \rightarrow back to $\sim 9 \text{ kN}$
= 14.9 double friction
= 7.5 kN friction

Recharged intensifier

Got to 600 MPa but leak at no 3.

572 MPa

LC 260

15g

- 6.64

12 MPa

release of oil p → ~~14.66~~ 14.66

Double friction ~ 10 kN

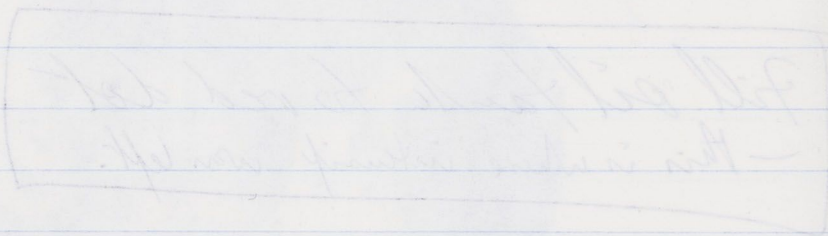
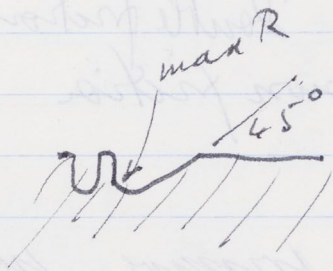
Down friction ~ 5 MPa

≡ 17 kN

On dropping pressure, no 3 leak stopped
at 505 MPa.

Fill oil tank to red dot
- this is where intensifier was left.

Con cracked at bottom of bottom ←
winding, dirty (overlap joint clean)



30/3/92

Furnace no. 001 returned for re-furbishing.

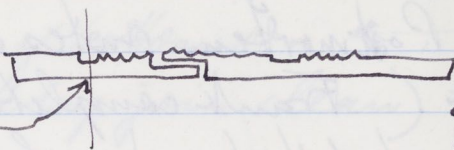
Post mortem notes in SERNO RIG NOTES book. Frank completed furnace & took to Ametek today for shipping. Details:

- retained upper part of core & windings; replaced lower part of core (this was early two-part core design) put in new lower core with winding, after modifying the grooves to have a 45° chamfer & better radius at bottom
- relieved the SS upper core at top to reduce chance of sticking
- extra Al_2O_3 paper layer over core, not up gap, to compensate for some shrinking of insulation which was re-used
- same upper & middle windings
- cleaned Az tubes on TK ends
- it assembled OK all connections OK
- MS inner tube in place at bottom

Furnace 005 returned Feb 1993
after accident to specimen.

Core smashed.

Earlier break



SS
upper
core
OK in
insulation

Lot of C on bottom winding

Needs: new core : Bending of TX 1000°C
or Coors

insulation OK - question of
changing to SAl

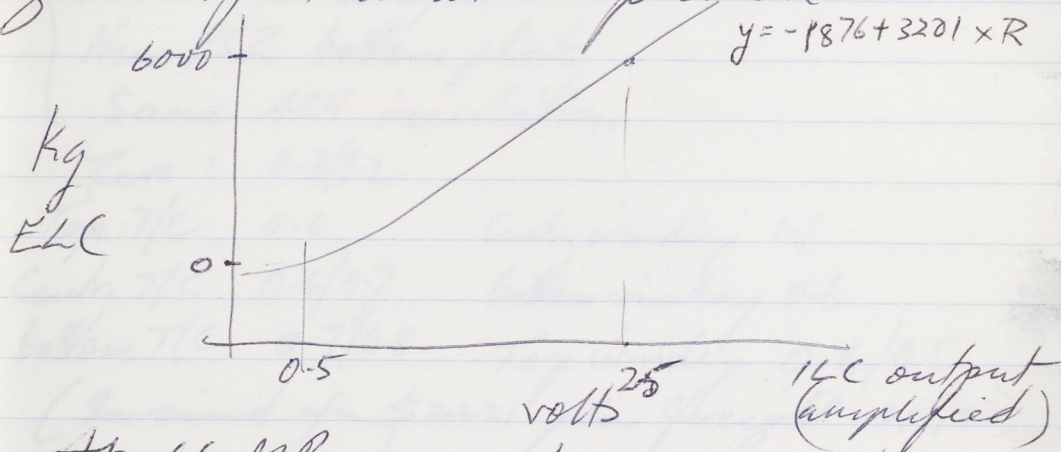
bottom Macor - several breaks.

Refurbished & sent back 29/3/93 with:

- 1) new Coors core & new PSZ upper core,
rewound
- 2) one new thermocouple
- 3) new bottom Macor piece

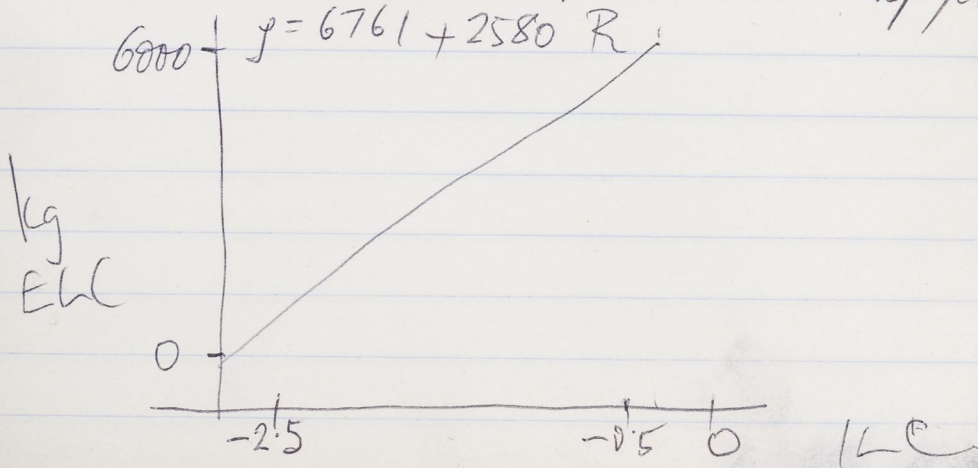
Montpellier 30/4/93

Test on re-wind load cell, now $\frac{1}{2}$ bridge with two strain gauges as other half bridge, & trimming to give approx zero output at atmos-pressure



At 66 MPa, zero has moved to $-2.6V$ (ie equiv to $-60 kN$). Previously it moved to $\sim +6V$ or thereabouts (about same load but opposite).

Now tried above calib at 66 MPa conf. pr.



Some hiccup at $\approx 0.6V$ (maybe wires)
Plot gives increased sensitivity at higher
pressure (unless this is due to friction?)

Furnace 005 Returned 30/7/93 after
blowout, for repair.

- { New core (Bundigs)
- { 3 new T/C complete
- { New PSZ bottom plate
- { Same ASH insulation.

Zero : 0.3/0.2

Top T/C 0.6 Centre winding 1.1

Centre T/C 0.6/0.7 Bottom winding 0.6

Bottom T/C 0.7/0.8 Top winding 0.6/0.5

(Invoiced for \$2221 plus freight 3/8/93).

THE REMAINING PAGES
IN THIS NOTEBOOK
ARE BLANK

